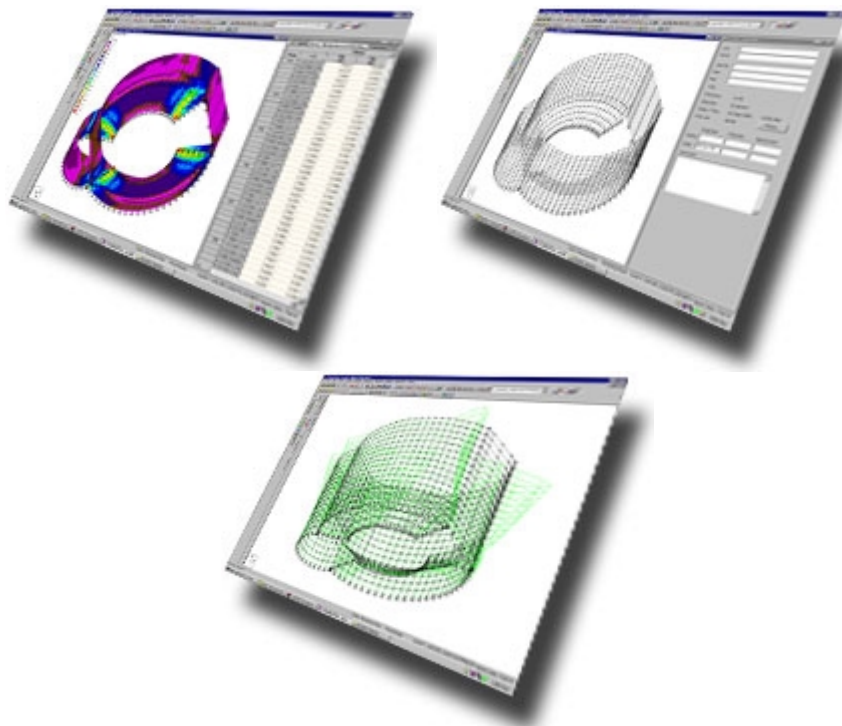


STAAD.*Pro*

NS 3472 / NPD



NS 3472 / NPD

September 2001

1994

STEEL DESIGN - CODE CHECK

DESIGN PARAMETER
THEORY MANUAL
TABULATED RESULTS

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1 Introduction

This user manual presents a description of the design basis, parameters and theory applied to STAAD.Pro for performing code checks according to NS 3472 ref. [1] and NPD ref. [5].

The code checks include:

- stability check (buckling)
- lateral buckling check

- yield check (von Mises)
- stability check including local plate buckling of un-stiffened pipe walls according to NPD

The code check is available for the following cross-section types:

- wide flange profiles (HEA, HEB, IPE etc.)
- pipe (OD xx ID xx)
- tube (RHS, HUP)
- channel
- angle type
- rectangular massive box (prismatic)
- user table (wide flange, I-sections, tapered I, tube, channel and RA angle)

The code check is not available for the following cross-section types:

- Double angles
- Tapered tubes
- Prismatic sections with too few section parameters defined
- Other sections that are not in the 'available' list above

Please note the following:

- NS 3472 and NPD code checking covered in this document are available through two separate STAAD.Pro Code check packages.
- This document is not a lecture in use of NS 3472 or NPD. This document explains how, and which parts of, the Norwegian steel codes that have been implemented in STAAD.Pro.
- When L-sections are used, the Code Check requires RA angle definition.
- Weld design is not included in the Norwegian code checks.
- The prismatic section defined in the code check (rectangular massive box) is not identical to the general prismatic profile defined in the STAAD.Pro analysis package.

EDR does not accept any liability for loss or damage from or in consequence for use of the program.

Nomenclature:

NS - refers to NS 3472 ref. [1]

NS2 - refers to NS 3472 ref. [6]

NPD - refers to NPD94 ref. [5]

2 Basis for code checking

2.1 General

This section presents general information regarding the implementation of the Norwegian codes of practice for structural steel design. This manual describes the procedures and theory used for both NS and NPD.

In general NS is used for all cross sections and shapes listed in section 1 of this manual. An exception is the treatment and check of pipe members in framed structures. NS does not give specific details about the treatment of pipes. Section 3.4 explains how this is adopted when NS is selected for code checking.

The NPD however have a more thorough check of pipe members, and consider the effect of local buckling of the pipe wall in conjunction with the stability check. In addition, the NPD code gives joint capacity formulae for brace to chord connections for pipe members.

The design philosophy and procedural logistics are based on the principles of elastic analysis and ultimate limit state design. Two major failure modes are recognized:

- failure by overstressing
- failure by stability considerations

The following sections describe the salient features of the design approach. Members are proportioned to resist the design loads without exceeding the characteristic stresses or capacities and the most economic section is selected on the basis of the least weight criteria. It is generally assumed that the user will take care of the detailing requirements like the provision of stiffeners and check the local effects like flange buckling, web crippling, etc.

The user is allowed complete control over the design process through the use of the parameters listed in Table 2.1. Default values of parameters will yield reasonable results in most circumstances. However, the user should control the design and verify results through the use of the design parameters.

2.2 Calculation of Forces and Bending Moments

Elastic analysis method is used to obtain the forces and moments for design. Analysis is done for the primary loading conditions and combinations provided by the user. The user is allowed complete flexibility in providing loading specifications and using appropriate load factors to create necessary load combinations.

2.3 Members with Axial Forces

For tension only members, axial tension capacity is checked for the ultimate limit stress. For compression members, axial compression capacity is checked in addition to lateral buckling and ultimate limit stress. The largest slenderness ratio (λ) shall not be greater than 250 according to NS 11.7 Stability is checked as per the procedure of NS 12.3. The buckling curves of NS fig. 3 have been incorporated into the STAAD.Pro code check. The coefficient α (as per NS Table 10) can be specified in both directions through the use of parameters CY and CZ. In the absence of parameters CY and /or CZ, default α - value will be according to NS table 11.

2.4 Members with Axial Force and Bending Moments

For compression members with bending, interaction formulae of NS table 12.3.4.2 are checked for appropriate loading situation. All compression capacities are calculated per the procedure of NS 12.3.

The equivalent moment factor β is calculated using the procedure of NS table 12. Two different approaches are used depending upon whether the members can sway or not. Conditions for sidesway and transverse loading can be specified through the use of parameters SSY and SSZ. For members that cannot sway, without transverse loading, coefficients β are calculated and proper dimensioning moments are used in the interaction formulae.

2.5 Lateral Buckling

Lateral torsional buckling is checked as per the procedure of NS 12.3.4. The procedure for calculation of ideal buckling moment for sections with two axis of symmetry has been implemented. The coefficient can be provided by the user through the use of parameter CB. In the absence of CB, a value of 1.0 will be used. Torsional properties for cross-sections (torsional constant and warping constant) are calculated using formulae from NS 3472. This results in slightly conservative estimates of torsional parameters. The program will automatically select the maximum moment in cases where M_{vd} is less than M_{zd} .

2.6 Von Mises Yield Criterion

Combined effect of axial, bending, horizontal/vertical shear and torsional shear stress is calculated at 13 sections on a member and up to 9 critical points at a section. The worst stress value is checked against yield stress divided by appropriate material factor.

The von Mises calculates as:

$$\sigma_j = \sqrt{(\sigma_x + \sigma_{by} + \sigma_{bz})^2 + 3(\tau_x + \tau_y + \tau_z)^2} \leq \frac{f_y}{\gamma_m}$$

2.7 Material Factor and nominal stresses

The design resistances are obtained by dividing the characteristic material strength by the material factor.

NS 3472: The material factor default value is 1.10. Other values may be input with the MF parameter. The nominal stresses should satisfy $\sigma_j \leq \frac{f_y}{\gamma_m} = f_d$

NPD: The general requirement is $\sum S(F_i \gamma_{fi}) \leq R_k / (\gamma_m \cdot \gamma_{mk}(S))$ according to NPD 3.1.1. For stability the NPD 3.1.1 and 3.1.3 requires that the structural coefficient is considered.

$$S_d \leq f_{kd} = \frac{f_k}{\gamma_m \cdot \gamma_{mk}(S_d)}$$

where

S_d	= reference stress or load effect resultant
f_k	= characteristic capacity
f_{kd}	= design capacity
γ_m	= material coefficient

γ_{mk} = structural coefficient

γ_m is default set to 1.10.

γ_{mk} shall be equal to 1.0 for frames. For pipe members γ_{mk} is a function of the reduced slenderness. In the STAAD.Pro implemented NPD code this is calculated automatically.

2.8 Code checking according to NPD

The following parts of Chapter 3 in the NPD guidelines have been implemented.

- a) Control of nominal stresses. (NPD 3.1.2).
- b) Buckling of pipe members in braced frames, including interaction with local shell buckling (NPD 3.2.2, 3.2.3).
- c) Buckling of un-stiffened closed cylindrical shells, including interaction with overall column buckling (NPD 3.4.4, 3.4.6, 3.4.7 and 3.4.9).
- d) Joint capacity check for gap as well as for overlap joints (NPD 3.5.2).

Check b) provides the unity check based on the beam-column buckling interaction formulae in NPD 3.2.2. The interaction between global and local buckling due to axial load and hydrostatic pressure is accounted for through computation of an axial characteristic capacity to replace the yield stress in the beam-column buckling formulae.

Note that check b) handles members subjected to axial loads, bending moments and hydrostatic pressure. In other words, check b) assumes that stresses resulting from shear and torsion are of minor importance, e.g. in jacket braces.

Check c) provides the unity check based on the stability requirement for un-stiffened cylindrical shells subjected to axial compression or tension, bending, circumferential compression or tension, torsion or shear. The unity check refers to the interaction formulae in NPD 3.4.4.1. The stability requirement is given in NPD 3.4.7.

2.9 Aluminium Check

STAAD.Pro performs stability check on aluminium alloys according to buckling curve in ECCS (European recommendation for aluminium alloy structures 1978). It is possible to select heat-treated or non heat-treated alloy from the parameter list in the STAAD.Pro input file.

For heat-treated use $CY=CZ=0.1590$, and for non heat-treated use $CY=CZ=0.2420$.

Tracks 1.0 and 9.0 print buckling curve H for heat-treated, and buckling curve N for non-heat-treated. The yield check is the same as for steel.

Table 2.1 Design parameters

STAAD.Pro STEEL DESIGN- CODE CHECK			
PARAMETER NAME	DEFAULT VALUE	DESCRIPTION	REFERENCE
BEAM	0.0 Note: must be set to 1.0	Parameter BEAM 1.0 ALL tells the program to calculate von Mises at 13 sections along each member, and up to 8 points at each section. (Depends on what kind of shape is used.)	Sec. NS 12.2.2
CODE	none	NS3472 for NS, NPD for NPD (NOR may also be used for both)	
CY CZ	Default see NS 3472	Buckling curve coefficient, α about local z-axis (strong axis). Represent the a, a_0 , b, c, d curve.	Fig. NS 3 Sec. NS 12.2 NS Table 11
BY	1.0	Buckling length coefficient, β , for weak axis buckling (y-y) (NOTE: BY > 0.0)	Fig. NS 3 Sec. NS 12.3
BZ	1.0	Buckling length coefficient, β , for strong axis buckling (z-z) (NOTE: BZ > 0.0)	Fig. NS 3 Sec. NS 12.3
FYLD	235	Yield strength of steel, f_y (St37) [N/mm ²]	Tab. NS 3
MF	1.1 (NS3472) 1.15 (NPD)	Material factor / Resistance factor, γ_m	Sec. NS 10.4.2 Sec. NPD 3.1
UNL	Member length	Effective length for lateral buckling calculations (specify buckling length). Distance between fork supports or between effective side supports for the beam	Sec. NS 12.3
CB	1.0	Lateral buckling coefficient, Ψ . Used to calculate the ideal buckling moments, M_{vi}	Sec. NS2 A5.5.2 Fig. NS2 A5.5.2a)-e)
SSY	0.0	0.0 = No sidesway. β calculated. > 0.0 = Sidesway in local y-axis weak axis $\beta=SSY$	Sec. NS 12.3.4 Tab. NS 12 Sec. NPD 3.2.1.4
SSZ	0.0	0.0 = No sidesway. β calculated. > 0.0 = Sidesway in local z-axis strong axis $\beta=SSY$	Sec. NS 12.3.4 Tab. NS 12 Sec NPD 3.2.1.4
CMY	1.0	Water depth in meters for hydrostatic pressure calculation for pipe members	Valid for the NPD code only
CMZ	0.49	α_{LT} for sections in connection with lateral buckling	Sec. NS 12.3.4 Fig. NS 6.

TRACK	0.0	0.0 = Suppress critical member stresses. 1.0 = Print all critical member stresses, i.e. DESIGN VALUES 2.0 = Print von Mises stresses. 9.0 = Large output, 1 page for each member. See section 7 and Appendix A for complete list of available TRACKs and print examples.	
RATIO	1.0	Permissible ratio of the actual to allowable stresses.	Sec. NS 12.3.4.2
DMAX	100.0 [cm]	Maximum allowable depth of steel section.	
DMIN	0.0 [cm]	Minimum allowable depth of steel section.	

The parameter CMY will, when given with negative value, define an inside pressure in pipe members. The pressure corresponds to given water depth in meters.

The parameter CB defines the ϕ value with respect to calculation of the ideal lateral buckling moment for single symmetric wide flange profiles, ref. NS app. 5.2.2.

EXAMPLE:
(Used at the end
of the input file)

```
* Code check according to NS3472
UNIT MMS NEWTON
PARAMETERS
CODE NS3472
BEAM 1.0 ALL
FYLD 340 ALL
MF 1.10 ALL
CY 0.49 MEMB 1
CZ 0.49 MEMB 1
BY 0.9 MEMB 1
BZ 0.7 MEMB 1
SSY 1.42 MEMB 1
SSZ 1.45 MEMB 1
CB 0.9 MEMB 1
RATIO 1.0 ALL
TRACK 9.0 ALL
UNIT KNS METER
LOAD LIST 1
CHECK CODE MEMB 1
FINISH
```


3 Stability check according to NS 3472

3.1 General description

The stability check is based on the assumption that both ends of the member are structural nodes. Buckling lengths and results for member with joints between the structural nodes have to be evaluated in each separate case.

Effects from local buckling or external hydrostatic pressure on pipes and tubes are not included.

The general stability criteria is: (ref. NS 12.3)

Buckling :

$$n_{\max} + k_z \cdot m_z + k_y \cdot m_y \leq 1$$

Lateral buckling :

$$\frac{n}{\chi_y} + k_{LT} \cdot \frac{m_z}{\chi_{LT}} + k_y \cdot m_y \leq 1$$

$$i = z, y \quad n_{\max} = \frac{n}{\chi_{\min}} \quad n = \frac{N_f}{N_d} \quad \chi_{\min} = \min(\chi_z, \chi_y) \quad \chi_i = \frac{N_{kd,i}}{N_d}$$

$$k_i = 1 - \mu_i \cdot \frac{n}{\chi_i \cdot \gamma_m} \leq 1.5 \quad \mu_i = \bar{\lambda}_i \cdot (2 \cdot \beta_{Mi} - 4) \leq 0.9$$

β_M ref. NS Tab. 12

$$k_{LT} = 1 - \mu_{LT} \cdot \frac{n}{\chi_y \cdot \gamma_m} \leq 1.0 \quad \mu_{LT} = 0.15 \cdot (\bar{\lambda}_y \cdot \beta_M - 1) \leq 0.9$$

$$\bar{\lambda}_i = \frac{\lambda_i}{\lambda_1} \quad \lambda_i = \frac{L_{ki}}{i_i} \quad \lambda_1 = \pi \cdot \sqrt{\frac{E}{f_y}}$$

$$\chi_i = \frac{1}{\phi + \sqrt{\phi^2 - (\bar{\lambda})^2}} \quad \phi = 0.5 \cdot \left[1 + \alpha \cdot (\bar{\lambda} - 0.2) + (\bar{\lambda})^2 \right]$$

α ref. NS Tab. 10 & 11

$$\chi_{LT} = \frac{1}{\phi_{LT} + \sqrt{\phi_{LT}^2 - (\bar{\lambda}_{LT})^2}}$$

$$\phi_{LT} = 0.5 \cdot \left[1 + \alpha \cdot (\bar{\lambda}_{LT} - 0.4) + (\bar{\lambda}_{LT})^2 \right] \quad \alpha \text{ ref. NS sec. 12.3.4.1}$$

$$\bar{\lambda}_{LT} = \sqrt{\frac{W_z \cdot f_z}{M_{cr}}} \quad M_{cr} = \psi \cdot M_{vio} \quad \psi \text{ ref. NS2 A5.5.2 a)-d)}$$

$$M_{vio} = \frac{\pi}{L} \cdot \sqrt{E \cdot I_z \cdot G \cdot I_T} \cdot \sqrt{1 + \frac{\pi^2}{L^2} \cdot \frac{E \cdot C_w}{G \cdot I_T}}$$

3.2 Determination of β_z and β_y

3.2.1 Lateral sidesway not prevented, NS 12.3

Code check input parameters SSZ and SSY must be given values > 0.0 for beams with movable end joints and beams with transverse loads between joints. The parameter equals the equivalent moment factor β ref. NS sec. 12.3.4.2 and Tab. 12.


$$\beta_y = \text{SSY}$$

$$\beta_z = \text{SSZ}$$

3.2.2 Lateral sidesway prevented, NS 12.3

Code check input parameters SSZ and SSY $\equiv 0$ for beams with unmovable end joints without transverse loads between joints. The equivalent moment factor β (for z and y) is calculated dependant on moment distributions as shown in Fig. 3.1.

Figure 3.1 β for different moment distributions

Moment diagram	$\beta_M (\beta_{LT})$
 <p style="text-align: center;">$-1 \leq \psi \leq 1$</p>	$\beta_{M, \psi} = 1,8 - 0,7 \psi$

3.3 Lateral buckling

The Ideal lateral buckling moment is calculated according to NS2 A5.5.2

$$M_{vi} = \psi \cdot M_{vio} = \psi \cdot 1,95 \frac{E}{L} \sqrt{I_y \cdot I_x} \sqrt{1 + \frac{\pi^2 \cdot 2,6 C_w}{L^2} \frac{C_w}{I_x}} \quad \text{concern double symmetric}$$

cross sections where ψ is given in NS fig. A5.5.2, (input parameter CB), L = member length for lateral buckling (input parameter UNL), C_w and I_x , see section 5. For single symmetric cross sections, the ideal lateral buckling moment is

$$M_{vix} = \phi \cdot \frac{\pi^2 E I_y}{L^2} \sqrt{\left(\frac{5a}{\pi^2} + \frac{r_x}{3} - y_s\right)^2 + C^2} - \left(\frac{5a}{\pi^2} + \frac{r_x}{3} - y_s\right) \quad \text{where}$$

$$C^2 = \frac{C_w + 0,039 L^2 I_T}{I_y} \quad \text{and } a \text{ is distance from profile CoG to point where the load}$$

is acting, assumed to be on top flange. The ϕ parameter (ref NS fig. A5.5.2.g) is controlled by the input parameter CB.

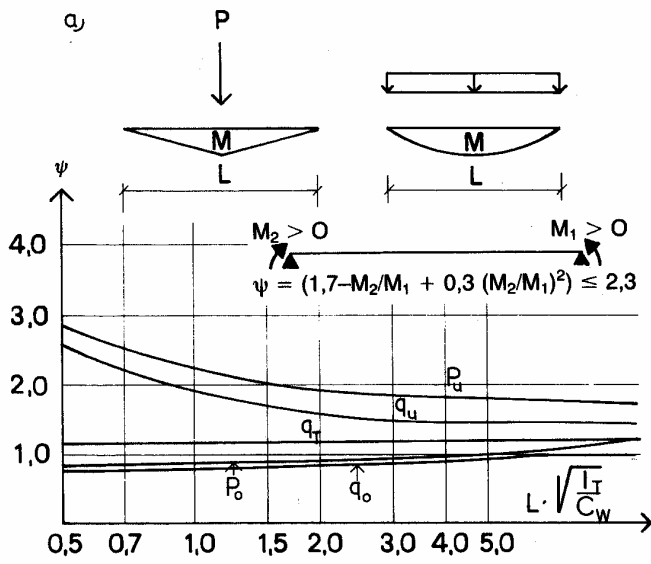


Fig. A5.5.2a ψ -koeffisienter for enkel bjelke

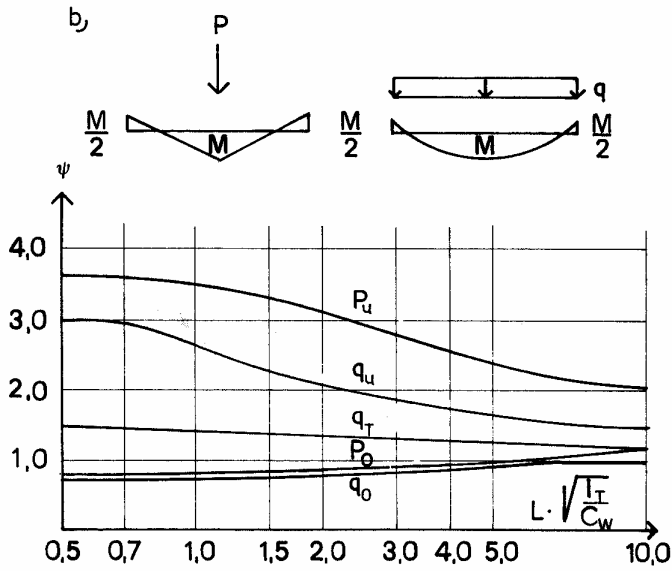


Fig. A5.5.2b ψ -koeffisienter for delvis innspent bjelke

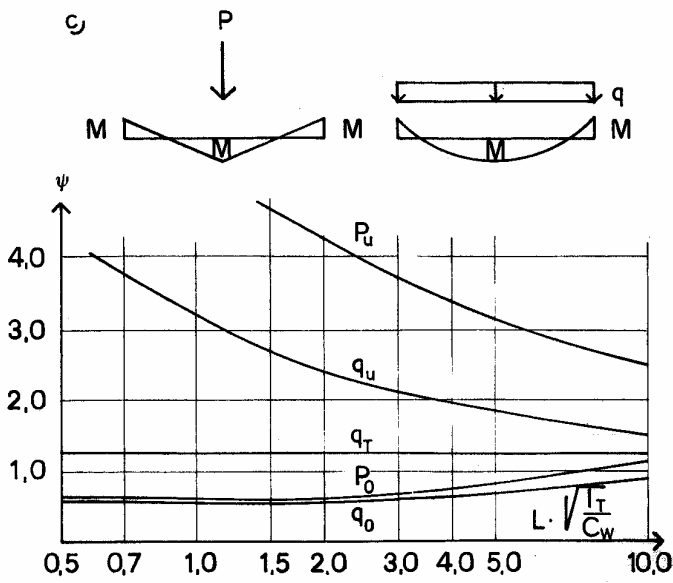


Fig. A5.5.2c ψ -koeffisienter for tilnærmet fullt innsente bjelker

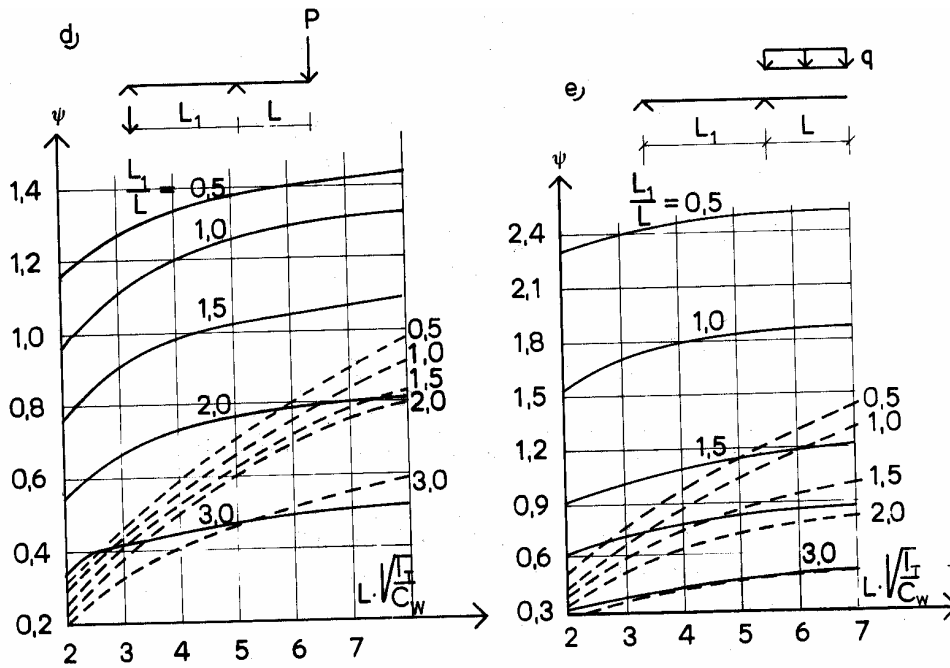


Fig. A5.5.2d og e ψ -koeffisienter for utragnet bjelke med enkel last og fordelt last. Stiplede kurver gjelder last på overflens.

3.4 Stability check of pipe members

The stability criteria applied for members with pipe cross section is:

$$IR = \frac{N}{N_{kd}} + \sqrt{\left(\frac{\bar{M}_z}{M_d \left(1 - \frac{N}{N_{Ezd}} \right)} \right)^2 + \left(\frac{\bar{M}_y}{M_d \left(1 - \frac{N}{N_{Eyd}} \right)} \right)^2} \leq 1.0$$

where

$$\frac{N}{N_{kd}} = \max \text{ of } \left(\frac{N}{N_{kzd}}, \frac{N}{N_{kyd}} \right)$$

\bar{M}_z and \bar{M}_y is given in NS 5.4.2.

For the print output option TRACK 9.0 $K_E \equiv 1.0$ and $M_{vd} \equiv M_d$

3.5 Angle profiles type RA (reverse angle)

The axial contribution to the total interaction ratio is checked according to the modified EECS-method, see NS A5.4.

The stability criterion is:

$$IR = \frac{N}{N_{kd}} + \frac{\bar{M}_y}{M_{yd} \left(1 - \frac{N}{N_{Eyd}}\right)} + \frac{\bar{M}_z}{M_{zd} \left(1 - \frac{N}{N_{Ezd}}\right)} \leq 1.0$$

$$\frac{N}{N_{kd}} = \max \text{ of } \left(\frac{N}{N_{kzd}}, \frac{N}{N_{kyd}} \right)$$

N_{kyd} and N_{kzd} are found from NS 3472 fig. 5.4.la C-curve

for y- and z-axis respectively.

$$\bar{\lambda}_{eff} = 0.60 + 0.57\bar{\lambda} \quad \text{for } \bar{\lambda} \leq \sqrt{2}$$

$$\bar{\lambda}_{eff} = \bar{\lambda} \quad \text{for } \bar{\lambda} > \sqrt{2}$$

$$\bar{\lambda} = \frac{\lambda_k}{\pi} \sqrt{\frac{f_y}{E}}$$

$$\lambda_k = \frac{l_k}{i}$$

$$i = \sqrt{\frac{I}{A}}$$

Possible lateral buckling effects and torsional buckling (NS A5.4.5) is not included in the code check. This has to be evaluated by the user separately.

3.6 Stability check of members with tapered section

Stability of members with tapered cross section is calculated as described in section 3.1. The cross section properties used in the formulae are calculated based on the average profile height. (I.e. I_z , I_y values are taken from the middle of the member.)

3.7 Lateral buckling for tension members

When compressive stress caused by large bending moment about strong axis is greater than tension stress from axial tension force, lateral buckling is considered as defined below.

$$\sigma_a = \frac{N}{A} \quad (+ \text{ tension } , - \text{ compression})$$

$$\sigma_{bz} = \frac{+ M_z}{W_z}$$

$$M_{\text{warp}} = 0 \quad \text{for } \sigma_a + \sigma_b \geq 0 \text{ (tension)}$$

$$M_{\text{warp}} = | \sigma_a + \sigma_{bz} | W_z \quad \text{for } \sigma_a + \sigma_b < 0 \text{ (compression)}$$

$$IR = \frac{M_{\text{warp}}}{M_{vd}} + \frac{M_{y,\text{max}}}{M_{yd}} \leq 1.0$$

4 Stability check according to NPD

4.1 Buckling of pipe members

Tubular beam-columns subjected to compression and lateral loading or end moments shall be designed in accordance with NPD 3.2.2

$$\sigma_c \gamma_{mk} + B \sigma_b^* + \sqrt{(B_z \sigma_{bz})^2 + (B_y \sigma_{by})^2} \leq \frac{f_y}{\gamma_m}$$

where

$$\sigma_c = \frac{N}{A} = \text{axial compressive stress}$$

γ_{mk} = structural coefficient

$$B = \text{bending amplification factor} = \frac{1}{1 - \mu}$$

B is the larger of B_z and B_y

B_z = bending amplification factor about the Z-axis

B_y = bending amplification factor about the Y-axis

$$\mu = \frac{\sigma_c}{f_E}$$

$$f_E = \frac{\pi^2 E}{l_k^2} i^2$$

$$i = \sqrt{\frac{I}{A}}$$

$$l_k = kl$$

k = effective length factor

$$\sigma_b^* = \sigma_c \left(\frac{f_y}{f_k} - 1 \right) \left(1 - \frac{f_k}{\gamma_m f_E} \right)$$

f_k = characteristic buckling capacity according to NS fig. 5.4.1a, curve A.

4.1.1 Interaction with local buckling, NPD 3.2.3

If the below conditions are not satisfied, the yield strength will be replaced with characteristic buckling stress given in NPD 3.4.

a) members subjected to axial compression and external pressure

$$\frac{d}{t} \leq 0,5 \sqrt{\frac{E}{f_y}}$$

b) members subjected to axial compression only

$$\frac{d}{t} \leq 0,1 \frac{E}{f_y}$$

4.2 Calculation of buckling resistance of cylinders

The characteristic buckling resistance is defined in accordance with NPD 3.4.4

$$f_k = \frac{f_y}{\sqrt{1 + \bar{\lambda}^4}}$$

where

$$\bar{\lambda}^2 = \frac{f_y}{\sigma_j} \left[\frac{\sigma_{a0}}{f_{ea}} + \frac{\sigma_{b0}}{f_{eb}} + \frac{\sigma_{p0}}{f_{ep}} + \frac{\tau}{f_{e\tau}} \right]$$

$$\sigma_j = \sqrt{(\sigma_a + \sigma_b)^2 - (\sigma_a + \sigma_b)\sigma_p + \sigma_p^2 + 3\tau^2}$$

$$\sigma_{a0} = \begin{cases} 0 & \sigma_a \geq 0 \\ -\sigma_a & \sigma_a < 0 \end{cases}$$

$$\sigma_{b0} = \begin{cases} 0 & \sigma_b \geq 0 \\ -\sigma_b & \sigma_b < 0 \end{cases}$$

$$\sigma_{p0} = \begin{cases} 0 & \sigma_p \geq 0 \\ \sigma_p & \sigma_p < 0 \end{cases}$$

σ_a = design axial stress in the shell due to axial forces (tension positive)

σ_b = design bending stress in the shell due to global bending moment (tension positive)

$\sigma_p = \sigma_\Theta$ = design circumferential stress in the shell due to external pressure (tension positive)

τ = design shear stress in the shell due to torsional moments and shear force.

f_{ea} , f_{eb} , f_{ep} and $f_{e\tau}$ are the elastic buckling resistances of curved panels or circular cylindrical shells subjected to axial compression forces, global bending moments, lateral pressure, and torsional moments and/or shear forces respectively.

4.3 Elastic buckling resistance for un-stiffened, closed cylinders

The elastic buckling resistance for un-stiffened closed cylinders according to NPD 3.4.6 is:

$$f_e = k \frac{\pi^2 E}{12(1-\nu^2)} \left(\frac{t}{l}\right)^2$$

where k is a buckling coefficient dependent on loading condition, aspect ratio, curvature, boundary conditions, and geometrical imperfections. The buckling coefficient is:

$$k = \psi \sqrt{1 + \left(\frac{p\zeta}{\psi}\right)^2}$$

The values of ψ , ζ and p are given in Table 4.1 for the most important loading cases.

Table 4.1 Buckling coefficients for un-stiffened cylindrical shells

	ψ	ζ	p
Axial stress	1	0,702 Z	$0,5 \left(1 + \frac{r}{150t}\right)^{-0,5}$
Bending	1	0,702Z	$0,5 \left(1 + \frac{r}{300t}\right)^{-0,5}$
Torsion and shear force	5,34	$0,856Z^{3/4}$	0,6
Lateral pressure	4	$1,04\sqrt{Z}$	0,6
Hydrostatic pressure	2	$1,04\sqrt{Z}$	0,6

The curvature parameter is defined by

$$Z = \frac{l^2}{rt} \sqrt{1-\nu^2}$$

For long shells the elastic buckling resistance against shear stresses is independent of shell

length. For cases with $\frac{l}{r} > 3,85\sqrt{\frac{r}{t}}$ the elastic buckling resistance may be taken as:

$$f_{et} = 0,25E \left(\frac{t}{r}\right)^{3/2}$$

For long shells and pressure vessels, the elastic buckling resistance against uniform lateral pressure is independent of length.

For cases with $\frac{l}{r} > 2,25\sqrt{\frac{r}{t}}$, the elastic buckling resistance may be taken as:

$$f_{ep} = 0,25E \left(\frac{t}{r}\right)^2$$

4.4 Stability requirements

The stability requirement for curved panels and un-stiffened cylindrical shells subjected to axial compression or tension, bending, circumferential compression or tension, torsion or shear is given by NPD 3.4.7:

$$\sigma_j \leq f_{kd}$$

where the design buckling resistance is

$$f_{kd} = \frac{f_k}{\gamma_m \cdot \gamma_{mk}}$$

4.5 Column buckling, NPD 3.4.9

For long cylindrical shells it is possible that interaction between shell buckling and overall column buckling may occur because second-order effects of axial compression alter the stress distribution as compared to that calculated from linear theory. It is necessary to take this effect into account in the shell buckling analysis when the reduced slenderness of the cylinder as a column exceeds 0,2 according to NPD 3.4.4.1.

σ_b shall be increased by an additional compressive stress which may be taken as:

$$\Delta\sigma = B\sigma_a \left(\frac{f_y}{f_k} - 1 \right) \left(1 - \frac{f_k}{f_e} \right) + (B - 1)\sigma_b$$

where

$$\bar{B} = \frac{1}{1 - \mu}$$

$$\bar{\lambda} = \sqrt{\frac{f_y}{f_e}}$$

$$f_e = \frac{\pi^2 E}{\lambda^2}$$

λ = slenderness of the cylinder as a column.

B , σ_a , σ_b and μ are calculated in accordance with NPD 3.2.2.

5 Yield check

The yield check is performed at member ends and at 11 equally spaced intermediate sections along the member length.

At each section the following forces are applied:

- F_X max. axial force along member
- F_Y actual shear in local y-direction at section
- F_Z actual shear in local z-direction at section
- M_X max. torsional moment along member
- M_Y actual bending about local y-axis at section
- M_Z actual bending about local z-axis at section

For all profiles other than angle sections absolute values of the stresses are used. For double symmetric profiles there will always be one stresspoint.

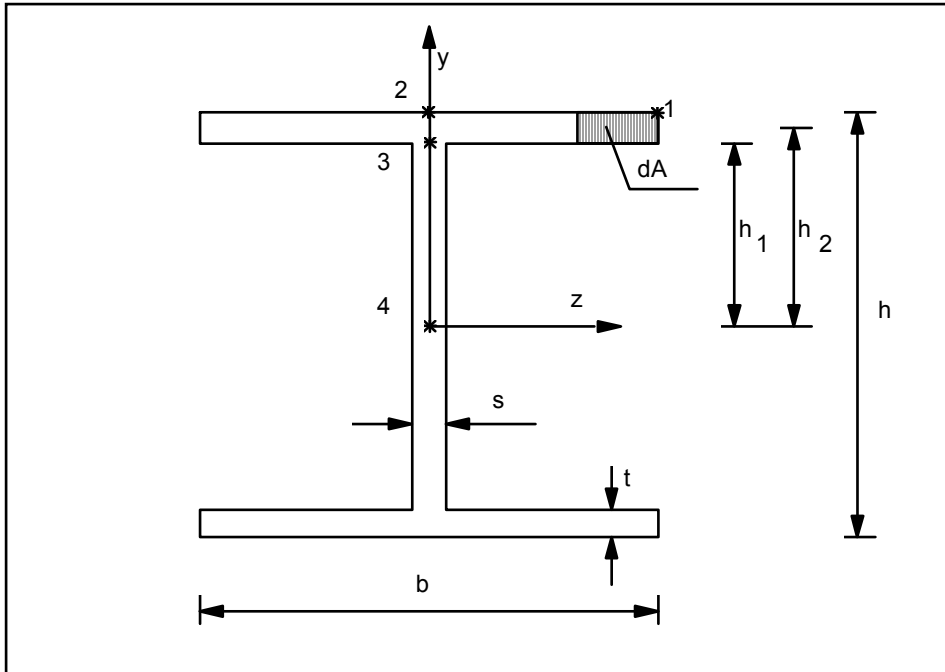
The stresses are calculated in several stress points at each member section. At each stress point the von Mises stress is checked as follows:

$$\sigma_j = \sqrt{\sigma_{tot}^2 + \sigma_p^2 - \sigma_{tot} \cdot \sigma_p + 3(\tau_x + \tau_y + \tau_z)^2} \leq \frac{f_y}{\gamma_m}$$

where $\sigma_{tot} = |\sigma_x + \sigma_{by} + \sigma_{bz}|$ and σ_p stress from hydrostatic pressure

5.1 Double symmetric wide flange profile

The von Mises stress is checked at 4 stress points as shown in figure below.



Section properties

A_x , I_x , I_y and I_z are taken from STAAD.Pro database

$$A_y = h \cdot s$$

Applied in STAAD.Pro print option PRINT MEMBER STRESSES

$$A_z = \frac{2}{3} b \cdot t \cdot 2$$

$$\tau_y = \frac{F_y}{A_y}, \tau_z = \frac{F_z}{A_z}$$

A_y and A_z are not used in the code check

$$C_w = \frac{(h-t)^2 b^3 t}{24}$$

ref. NS app. C3

$$T_y = dA \cdot z$$

$$T_z = dA \cdot y$$

General stress calculation

$$\sigma = \sigma_x + \sigma_{by} + \sigma_{bz} = \frac{F_x}{A_x} + \frac{M_y}{I_y} z + \frac{M_z}{I_z} y$$

$$\tau = \tau_x + \tau_y + \tau_z = \frac{M_x}{I_x} c + \frac{V_y T_z}{I_z t} + \frac{V_z T_y}{I_y t}$$

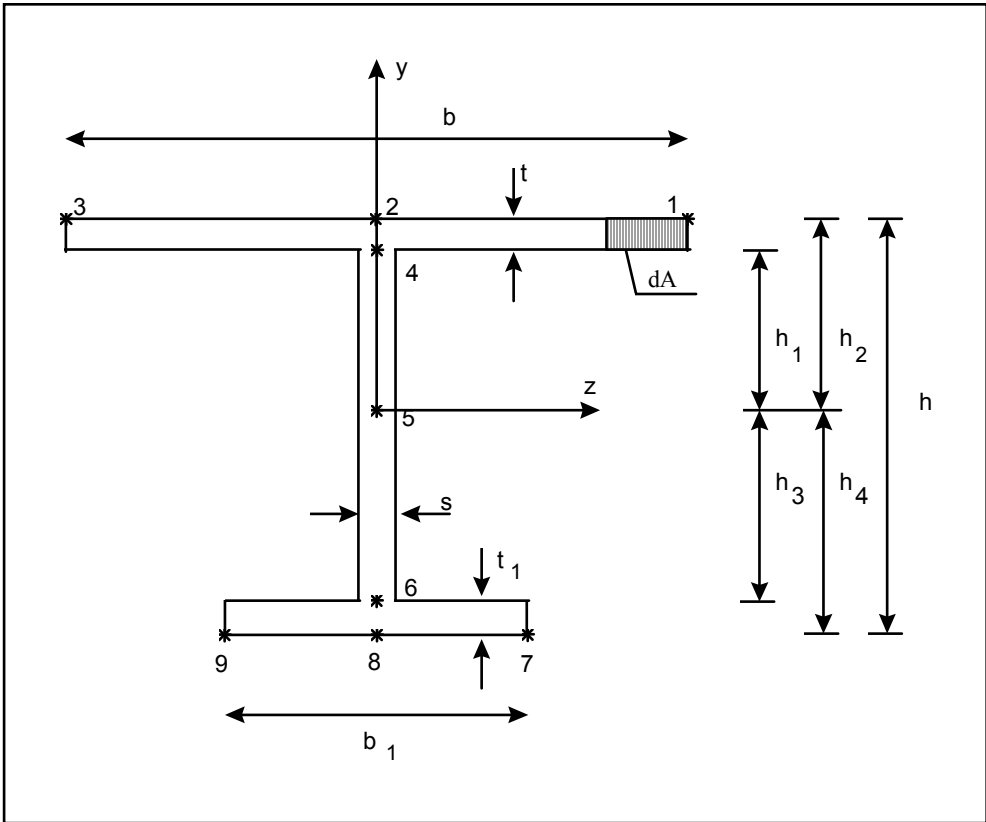
Stress calculation at selected stress points

Point no.	σ_x	σ_{by}	σ_{bz}	τ_x	τ_y	τ_z
1	$\frac{F_x}{A_x}$	$\frac{M_y}{I_y} \cdot \frac{b}{2}$	$\frac{M_z}{I_z} \cdot \frac{h}{2}$	$\frac{M_x}{I_x} t$	0	0
2	$\frac{F_x}{A_x}$	0	$\frac{M_z}{I_z} \cdot \frac{h}{2}$	$\frac{M_x}{I_x} t$	$\frac{F_y}{I_z} \frac{bth_2}{2t}$	$\frac{F_z}{I_y} \frac{tb^2}{8t}$
3	$\frac{F_x}{A_x}$	0	$\frac{M_z}{I_z} h_1$	$\frac{M_x}{I_x} s$	$\frac{F_y}{I_z} \frac{bth_2}{s}$	0
4	$\frac{F_x}{A_x}$	0	0	$\frac{M_x}{I_x} s$	$\frac{F_y}{I_z} \frac{(bth_2 + 0.5h_1^2s)}{s}$	0

In general wide flange profiles are not suitable for large torsional moments. The reported torsional stresses are indicative only. For members with major torsional stresses a separate evaluation has to be carried out. Actual torsional stress distribution is largely dependent on surface curvature at stress point and warping resistance.

5.2 Single symmetric wide flange profile and tapered section

The von Mises stress is checked at 9 stress points as shown in figure below.



Section properties

A_x , I_x , I_y and I_z are taken from STAAD.Pro database, except for tapered sections where these values are calculated for each section checked. (I.e. I_z , I_y values are taken from the middle of the member.)

$A_y = h \cdot s$ Applied in STAAD.Pro print option PRINT MEMBER STRESSES

$$A_z = \frac{2}{3}(b \cdot t + b_1 \cdot t_1) \quad \tau_y = \frac{F_y}{A_y}, \tau_z = \frac{F_z}{A_z}$$

A_y and A_z are not used in the code check

$$C_w = \frac{b^3 t \cdot b_1^3 t_1 \cdot (h - t / 2 - t_1 / 2)^2}{12(b^3 t + b_1^3 t_1)} \quad \text{ref. NS app. C3}$$

$$T_y = dA \cdot z$$

$$T_z = dA \cdot y$$

General stress calculation

$$\sigma = \sigma_x + \sigma_{by} + \sigma_{bz} = \frac{F_x}{A_x} + \frac{M_y}{I_y} z + \frac{M_z}{I_z} y$$

$$\tau = \tau_x + \tau_y + \tau_z = \frac{M_x}{I_x} c + \frac{V_y T_z}{I_z t} + \frac{V_z T_y}{I_y t}$$

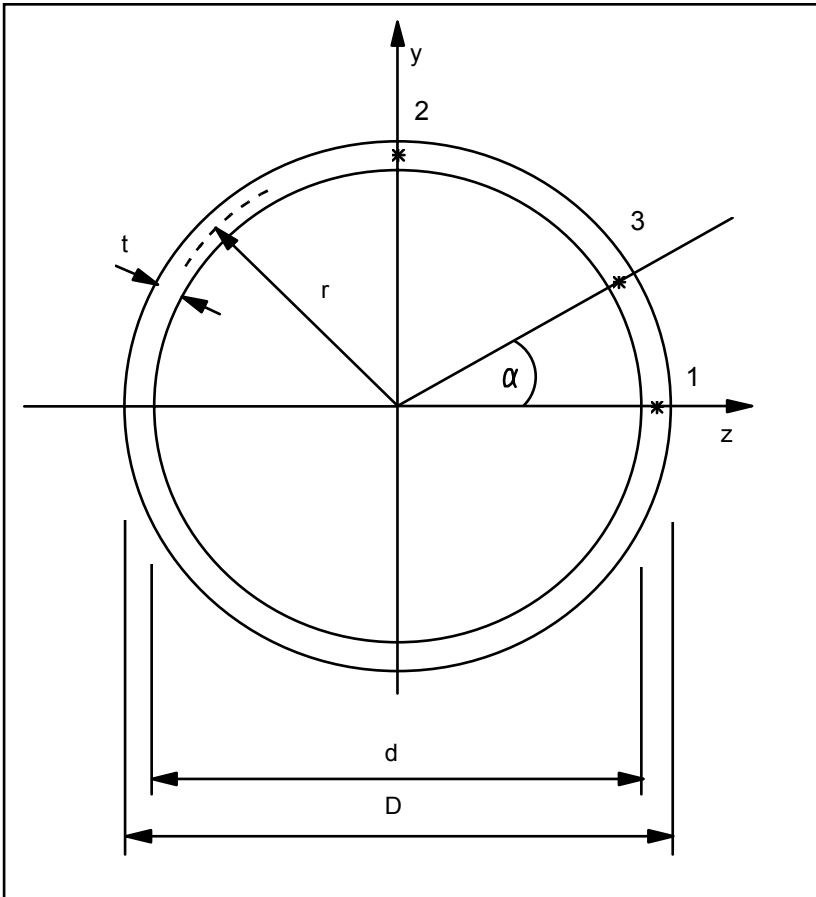
Stress calculation at selected stress points

Point no.	σ_x	σ_{by}	σ_{bz}	τ_x	τ_y	τ_z
1	$\frac{F_x}{A_x}$	$-\frac{M_y}{I_y} \frac{b}{2}$	$\frac{M_z}{I_z} h_2$	$\frac{M_x}{I_x} t$	0	0
2	$\frac{F_x}{A_x}$	0	$\frac{M_z}{I_z} h_2$	$\frac{M_x}{I_x} t$	$\frac{F_y}{I_z} \frac{bt(h_1+t/2)}{2t}$	$\frac{F_z}{I_y} \frac{tb^2}{8t}$
3	$\frac{F_x}{A_x}$	$\frac{M_y}{I_y} \frac{b}{2}$	$\frac{M_z}{I_z} h_2$	$\frac{M_x}{I_x} t$	0	0
4	$\frac{F_x}{A_x}$	0	$\frac{M_z}{I_z} h_1$	$\frac{M_x}{I_x} s$	$\frac{F_y}{I_z} \frac{bt(h_1+t/2)}{s}$	0
5	$\frac{F_x}{A_x}$	0	0	$\frac{M_x}{I_x} s$	$\frac{F_y}{I_z} \frac{(bt(h_1+t/2)+0.5h_1^2s)}{s}$	0
6	$\frac{F_x}{A_x}$	0	$-\frac{M_z}{I_z} h_3$	$\frac{M_x}{I_x} s$	$\frac{F_y}{I_z} \frac{b_1t_1(h_3+t_1/2)}{s}$	0
7	$\frac{F_x}{A_x}$	$-\frac{M_y}{I_y} \frac{b_1}{2}$	$-\frac{M_z}{I_z} h_4$	$\frac{M_x}{I_x} t_1$	0	0
8	$\frac{F_x}{A_x}$	0	$-\frac{M_z}{I_z} h_4$	$\frac{M_x}{I_x} t_1$	$\frac{F_y}{I_z} \frac{b_1t_1(h_3+t_1/2)}{2t_1}$	$\frac{F_z}{I_y} \frac{t_1b^2}{8t_1}$
9	$\frac{F_x}{A_x}$	$\frac{M_y}{I_y} \frac{b_1}{2}$	$-\frac{M_z}{I_z} h_4$	$\frac{M_x}{I_x} t_1$	0	0

In general wide flange profiles are not suitable for large torsional moments. The reported torsional stresses are indicative only. For members with major torsional stresses a separate evaluation has to be carried out. Actual torsional stress distribution is largely dependent on surface curvature at stress point and warping resistance.

5.3 Pipe profile

The von Mises stress is checked in 3 stress points as shown in figure below.



Section properties

$$d = D - 2t$$

$$r = 0.5 (D - t)$$

$$\alpha = \tan^{-1} \frac{M_z}{M_y}$$

$$A_x = \frac{\pi}{4} (D^2 - d^2)$$

$$A_y = A_z = 0.5A_x$$

$$I_x = 2I_z = \frac{\pi}{32} (D^4 - d^4)$$

$$I_y = I_z = \frac{\pi}{64} (D^4 - d^4)$$

Note!

In the STAAD.Pro analysis package slightly different values are used for A_y , A_z and I_x , however this has insignificant influence on the force distribution.

$$A_Y = A_Z = 0.6A_x$$

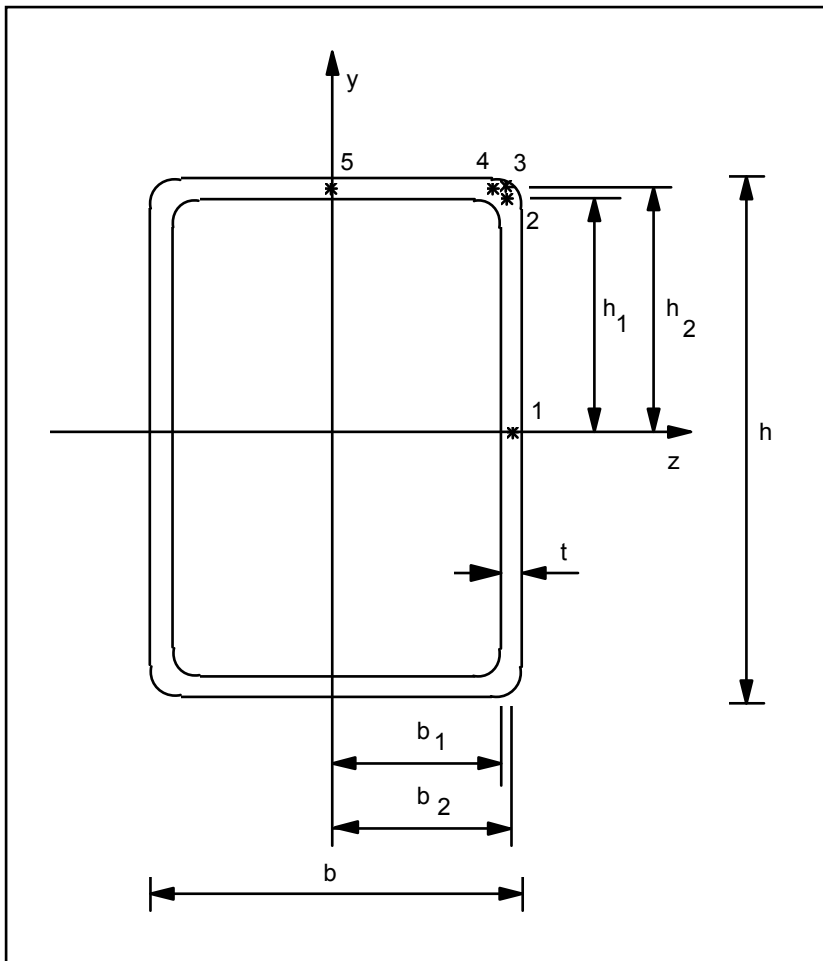
$$I_x = 2\pi R^3 t$$

Stress calculation at selected stress points

Point no.	σ_x	σ_{by}	σ_{bz}	τ_x	τ_y	τ_z
1	$\frac{F_x}{A_x}$	$\frac{M_y \cdot r}{I_y}$	0	$\frac{M_T \cdot r}{I_x}$	$\frac{F_y}{0.5A_x}$	0
2	$\frac{F_x}{A_x}$	0	$\frac{M_z \cdot r}{I_z}$	$\frac{M_T \cdot r}{I_x}$	0	$\frac{F_z}{0.5A_x}$
3	$\frac{F_x}{A_x}$	$\sigma_b = \frac{\sqrt{M_y^2 + M_z^2}}{I_z} r$		$\frac{M_T \cdot r}{I_x}$	$\tau = \frac{\sqrt{F_y^2 + F_z^2}}{0.5A_x}$	

5.4 Tube profile

Tube sections are rectangular or quadratic hollow uniform profiles. Critical stress is checked at 5 locations as shown in figure below.



Section properties

A_x , I_x , I_y and I_z are taken from STAAD.Pro database.

$A_y = 2ht$ Similar as for wide flange profiles, see sec. 5.2

$A_z = 2 \cdot \frac{2}{3} bt$ A_y and A_z are not used in code checks.

$$C_w = \frac{b^2 h^2 t (h-b)^2}{24 (h+b)} \text{ ref. NS app. C3.}$$

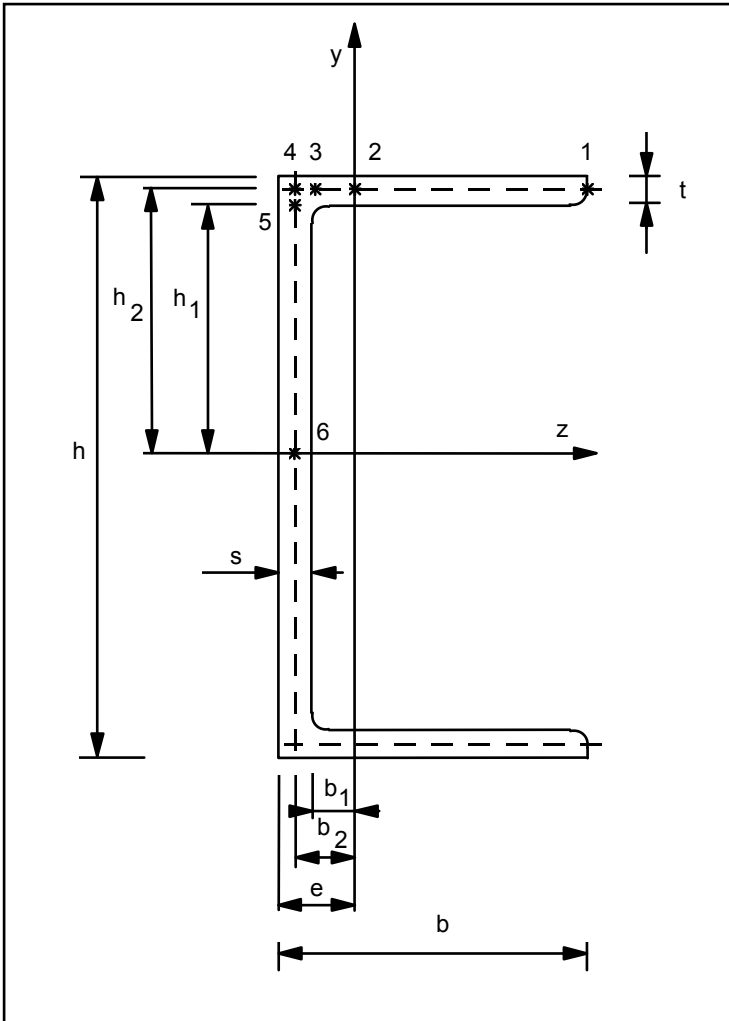
Stress calculation at selected stress points

Point no.	σ_x	σ_{by}	σ_{bz}	τ_x	τ_y	τ_z
1	$\frac{F_x}{A_x}$	$\frac{M_y}{I_y} b_2$	0	$\frac{M_x(h-t)(b-t)}{I_x(h+b-2t)}$	$\frac{F_y}{I_z} \frac{bth_2 + th_1^2}{2t}$	0
2	$\frac{F_x}{A_x}$	$\frac{M_y}{I_y} b_2$	$\frac{M_z}{I_z} h_1$	$\frac{M_x(h-t)(b-t)}{I_x(h+b-2t)}$	$\frac{F_y}{I_z} \frac{bth_2}{2t}$	$\frac{F_z}{I_y} \frac{2h_1tb_2}{2t}$
3	$\frac{F_x}{A_x}$	$\frac{M_y}{I_y} b_2$	$\frac{M_z}{I_z} h_2$	$\frac{M_x(h-t)(b-t)}{I_x(h+b-2t)}$	$\frac{F_y}{I_z} \frac{2b_2th_2}{2t}$	$\frac{F_z}{I_y} \frac{2h_2tb_2}{2t}$
4	$\frac{F_x}{A_x}$	$\frac{M_y}{I_y} b_1$	$\frac{M_z}{I_z} h_2$	$\frac{M_x(h-t)(b-t)}{I_x(h+b-2t)}$	$\frac{F_y}{I_z} \frac{2b_1th_2}{2t}$	$\frac{F_z}{I_y} \frac{htb_2}{2t}$
5	$\frac{F_x}{A_x}$	0	$\frac{M_z}{I_z} h_2$	$\frac{M_x(h-t)(b-t)}{I_x(h+b-2t)}$	0	$\frac{F_z}{I_y} \frac{htb_2 + tb_1^2}{2t}$

The general stress formulation is given in sec. 5.2.

5.5 Channel profile

For channel profiles the von Mises stress is checked at 6 locations as shown in figure below.



Cross section properties

A_x , S_y , S_z , I_x , I_y and I_z are taken from STAAD.Pro database.

$A_y = 2ht$ Similar as for wide flange profiles,
see sec. 5.2

$A_z = 2 \cdot \frac{2}{3} bt$ A_y and A_z are not used in code checks

$$e = b - \frac{I_y}{S_y}$$

$$x = h - t$$

$$y = b - \frac{s}{2}$$

$$C_w = \frac{x^2 y^3 t}{12} \frac{(2xs + 3yt)}{(xs + 6yt)} \text{ ref. [4] tab. 21, case 1}$$

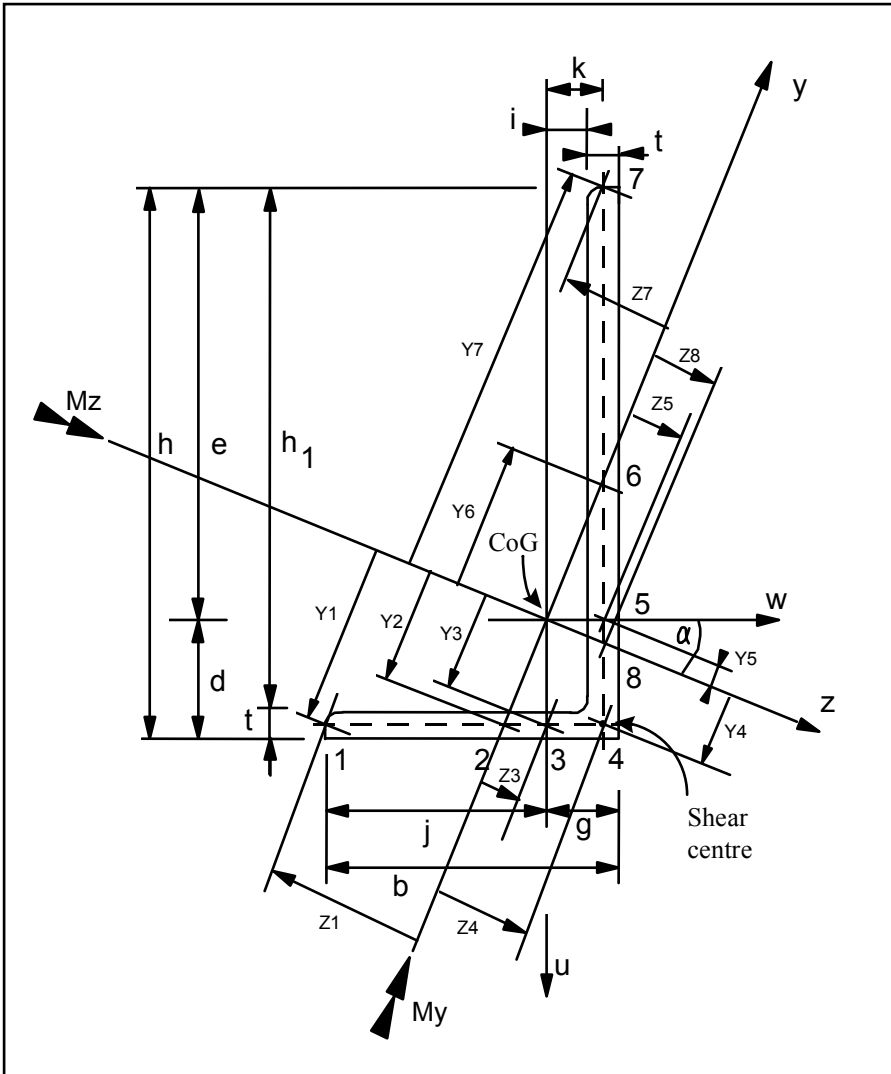
Stress calculations at selected stress points.

Point no.	σ_x	σ_{by}	σ_{bz}	τ_x	τ_y	τ_z
1	$\frac{F_x}{A_x}$	$\frac{M_y}{I_y} (b-e)$	$\frac{M_z}{I_z} h_2$	$\frac{M_x}{I_x} t$	0	0
2	$\frac{F_x}{A_x}$	0	$\frac{M_z}{I_z} h_2$	$\frac{M_x}{I_x} t$	$\frac{F_y}{I_z} \frac{(b-e)t \cdot h_2}{t}$	$\frac{F_z}{I_y} \frac{0.5(b-e)^2 t}{t}$
3	$\frac{F_x}{A_x}$	$\frac{M_y}{I_y} b_1$	$\frac{M_z}{I_z} h_2$	$\frac{M_x}{I_x} t$	$\frac{F_y}{I_z} \frac{(b-s)th_2}{t}$	$\frac{F_z}{I_y} \frac{(b-s)t\{0.5(b+s)-e\}}{t}$
4	$\frac{F_x}{A_x}$	$\frac{M_y}{I_y} b_2$	$\frac{M_z}{I_z} h_2$	$\frac{M_x}{I_x} t$	$\frac{F_y}{I_z} \frac{(b-0.5s)th_2}{t}$	$\frac{F_z}{I_y} \frac{\left(b-\frac{s}{2}\right)t\left\{0.5\left(b+\frac{s}{2}\right)-e\right\}}{t}$
5	$\frac{F_x}{A_x}$	$\frac{M_y}{I_y} b_2$	$\frac{M_z}{I_z} h_1$	$\frac{M_x}{I_x} s$	$\frac{F_y}{I_z} \frac{bth_2}{s}$	$\frac{F_z}{I_y} \frac{bt\left(\frac{b}{2}-e\right)}{s}$
6	$\frac{F_x}{A_x}$	$\frac{M_y}{I_y} b_2$	0	$\frac{M_x}{I_x} s$	$\frac{F_y}{I_z} \frac{bth_2 + 0.5sh_1^2}{s}$	0

The general stress formulation is given in sec. 5.2.

5.6 Angle profile type RA (reverse angle)

For angle profiles the von Mises check is checked at 8 stress points as shown in figure below.



Axes y and z are principal axes.

Axes u and w are local axes.

Cross section properties

A_x , I_x , I_y and I_z are taken from STAAD.Pro database

$$A_y = \frac{2}{3} ht \quad \text{Applied in STAAD.Pro print option PRINT MEMBER STRESSES}$$

$$A_z = \frac{2}{3} bt \quad \tau_y = \frac{F_x}{A_y}, \tau_z = \frac{F_z}{A_z}$$

and A_z are not used in the code check.

$$h_2 = 0.5 h_1 + t$$

$$f = h_1 - e$$

$$d = \frac{th_1h_2 + 0.5t^2b}{A_x}$$

$$g = \frac{t^2h_1 + tb^2}{2A_x}$$

$$I_u = \frac{h_1t^3}{12} + htk^2 + \frac{tb^3}{12} + tb\left(\frac{b}{2} - g\right)^2$$

$$I_w = \frac{th_1^3}{12} + h_1t(h_2 - d)^2 + \frac{bt^3}{12} + bt(d - 0.5t)^2$$

$$I_{uw} = \frac{\left(d - \frac{1}{2}\right)t}{2}(g^2 - j^2) - \frac{kt}{2}(e^2 - f^2)$$

$$\alpha = 0.5 \tan^{-1}\left(\frac{2I_{uw}}{I_u - I_w}\right)$$

Section forces

The section forces from the STAAD.Pro analysis are about the principle axis y and z.
The second moment of area (T_y & T_z):

$$T_y = A Z$$

$$T_z = A Y$$

Stress calculation at selected stress points

Point no.	σ_x	σ_{by}	σ_{bz}	τ_x	τ_y	τ_w
1	$\frac{F_x}{A_x}$	$-\frac{M_y Z1}{I_y}$	$-\frac{M_z Y1}{I_z}$	$\frac{M_x t}{I_x}$	0	0
2	$\frac{F_x}{A_x}$	0	$-\frac{M_z Y2}{I_z}$	$\frac{M_x t}{I_x}$	$\frac{F_y T_z}{I_z t}$	$\frac{F_z T_y}{I_y t}$
3	$\frac{F_x}{A_x}$	$\frac{M_y Z3}{I_y}$	$-\frac{M_z Y3}{I_z}$	$\frac{M_x t}{I_x}$	$\frac{F_y T_z}{I_z t}$	$\frac{F_z T_y}{I_y t}$
4	$\frac{F_x}{A_x}$	$\frac{M_y Z4}{I_y}$	$-\frac{M_z Y4}{I_z}$	$\frac{M_x t}{I_x}$	$\frac{F_y T_z}{I_z t}$	$\frac{F_z T_y}{I_y t}$
5	$\frac{F_x}{A_x}$	$\frac{M_y Z5}{I_y}$	$\frac{M_z Y5}{I_z}$	$\frac{M_x t}{I_x}$	$\frac{F_y T_z}{I_z t}$	$\frac{F_z T_y}{I_y t}$
6	$\frac{F_x}{A_x}$	0	$\frac{M_z Y6}{I_z}$	$\frac{M_x t}{I_x}$	$\frac{F_y T_z}{I_z t}$	$\frac{F_z T_y}{I_y t}$
7	$\frac{F_x}{A_x}$	$-\frac{M_y Z7}{I_y}$	$\frac{M_z Y7}{I_z}$	$\frac{M_x t}{I_x}$	$\frac{F_y T_z}{I_z t}$	$\frac{F_z T_y}{I_y t}$
8	$\frac{F_x}{A_x}$	$\frac{M_y Z8}{I_y}$	0	$\frac{M_x t}{I_x}$	0	0

An additional torsional moment is calculated based on:

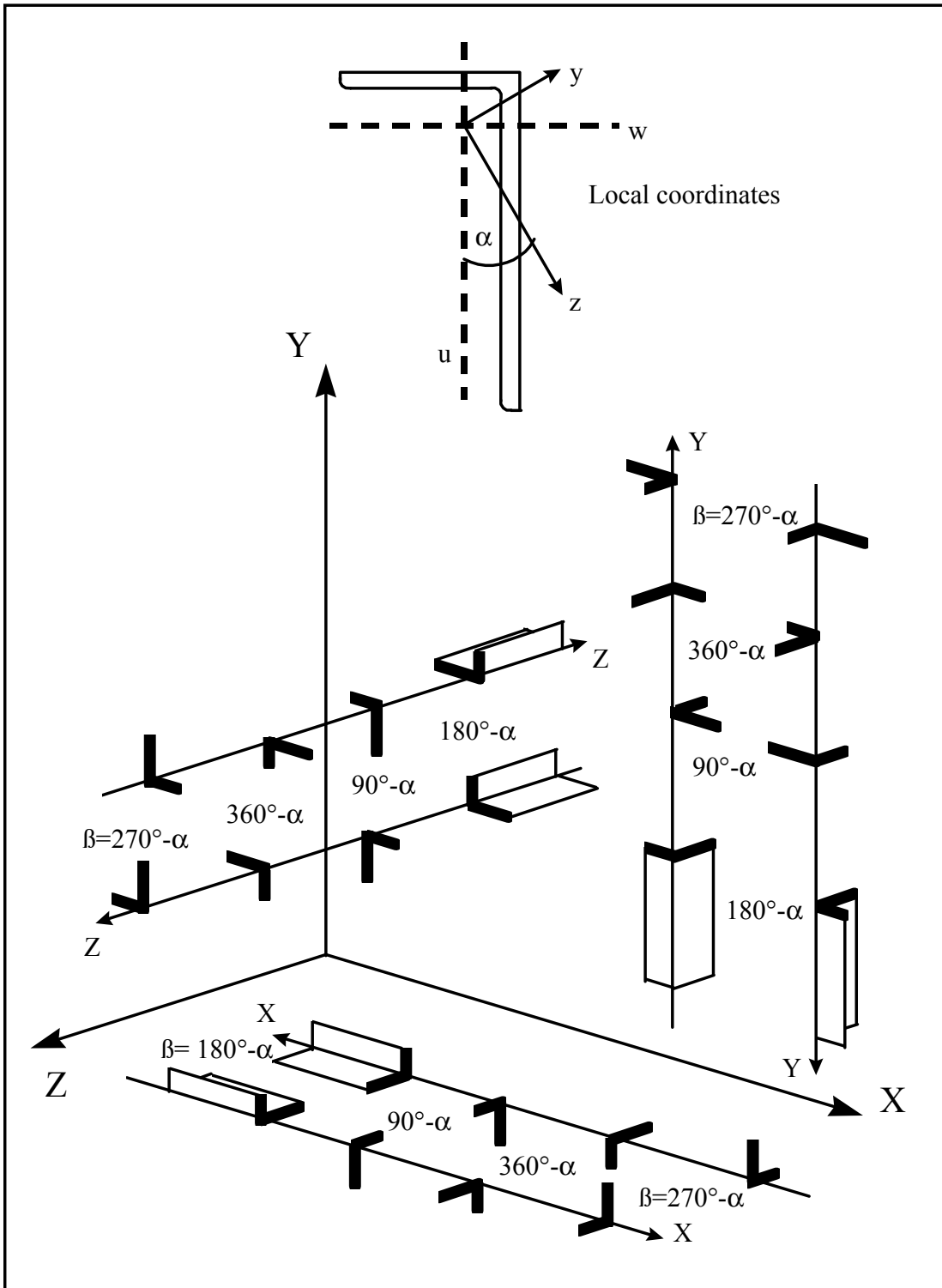
$$M_T = F_y Z4$$

$$M_T = F_z Y4$$

This torsion moment is included in M_x if F_y and F_z exist.

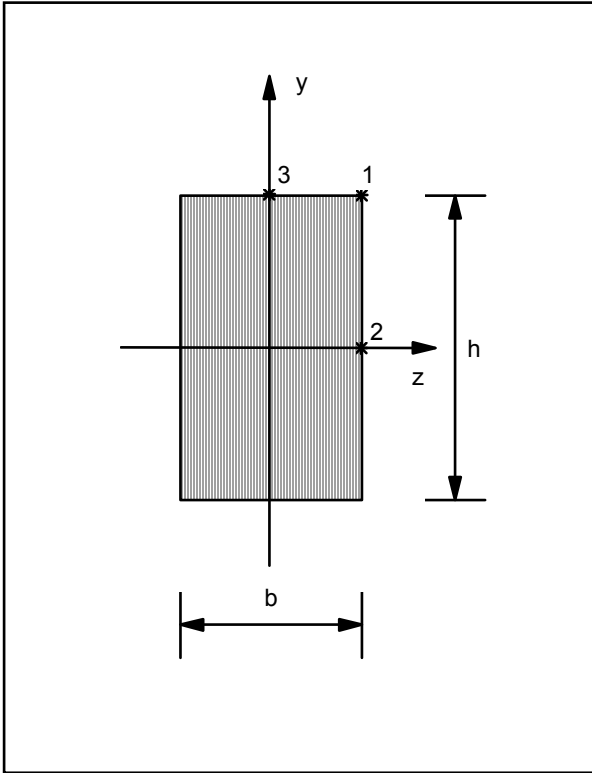
Beta-rotation of equal & unequal legged angles

(Note: the order of the joint numbers in the member incidence command specifies the direction of the local x-axis.)



5.7 Rectangular massive box (prismatic)

Code check of the general purpose prismatic cross section defined in the STAAD.Pro analysis package is not available. The prismatic section is assumed to be a rectangular massive box and the von Mises stress is checked at 3 locations as shown in figure below.



Note that 'b' may not be much greater than 'h'.

If that is the case, define the member with $h > b$ and Beta angle 90° instead.

Section properties

A_x , A_y , A_z , I_x , I_y , I_z , b and h are given by the user, see STAAD.Pro Reference Manual, sec. 5.19.2

($b = ZD$, $h = YD$)

$$C_w = \frac{1}{24} \frac{b (h-b)^2}{h+b} h^2 b^2 \quad \text{ref. NS app. C3.}$$

General stress calculation

$$\sigma = \sigma_x + \sigma_{by} + \sigma_{bz} = \frac{F_x}{A_x} + \frac{M_y}{I_y} z + \frac{M_z}{I_z} y$$

$$\tau = \tau_x + \tau_y + \tau_z = \tau_{x,\max} \left(\frac{c}{b} \right)^2 + \frac{V_y}{A_y} + \frac{V_z}{A_z}$$

$$\tau_{x,\max} = \frac{M_x(1.5h + 0.9b)}{0.5h^2 b^2}$$

ref. [4] tab. 20, case 4 at midpoint the largest side i.e. point 2

Stress calculation at selected stress points

Point no.	σ_x	σ_{by}	σ_{bz}	τ_x	τ_y	τ_z
1	$\frac{F_x}{A_x}$	$\frac{M_y}{I_y} \frac{b}{2}$	$\frac{M_z}{I_z} \frac{h}{2}$	$\tau_{x,\max} \frac{b^2}{b^2 + h^2}$	0	0
2	$\frac{F_x}{A_x}$	$\frac{M_y}{I_y} \frac{b}{2}$	0	$\tau_{x,\max}$	$\frac{F_y}{A_y}$	0
3	$\frac{F_x}{A_x}$	0	$\frac{M_z}{I_z} \frac{h}{2}$	$\tau_{x,\max} \frac{b^2}{h^2}$	0	$\frac{F_z}{A_z}$

6 Tubular joint check, NPD 3.5

For pipe members, punching shear capacity is checked in accordance with the NPD sections 3.5.1 to 3.5.2, except 3.5.2.4. The chord is defined as the member with the greater diameter in the joint. If the diameters are the same the programme selects the member with the greater thickness of the two. The chord members must be collinear by 5 degrees.

The punching shear run sequence is performed in two steps. The programme will first identify all tubular joints and classify them as T type joints (TRACK99). The joints to be checked will be listed in a file specified in the CODE NPD parameter list, below called GEOM1. This file is used as input in the second run. The file is an editable ASCII file saved under the file name given in the CODE NPD parameter. The TRACK parameter is then set to 98 which directs the program to read from the file GEOM1 file and use it as input to the second run, i.e. the joint capacity checking. The programme will check the capacity for both chord members entering the joint. The local y and z moments will be transformed into the plane defined by the joint itself and the far end joints of the brace and chord, defined as in- and out-of plane moments.

The ASCII file should be edited to reflect the correct classification of the joints, gap, can or stub dimensions, yield stress and other geometric options if required. The programme will not change the brace or chord definition if this is changed or modified in the input file GEOM1. See Appendix A page xx for GEOM1 example file.

Joint classification parameters in the file GEOM1 are:

KO K joint overlapped
 KG K joint with gap
 TY T or Y joint
 X X joint

Input example for the classification run.

```
*CLASSIFICATION OF JOINTS, TRACK 99
UNITS MM NEWTON
PARAMETER
CODE NPD GEOM1
FYLD 350 ALL
TRACK 99 ALL
BEAM 1.0 ALL
CHECK CODE ALL
```

6.1 Static strength of tubular joints

The basic consideration is the chord strength. The required chord wall thickness shall be determined when the other dimensions are given.

The following symbols are used:

T = Cord wall thickness
 t = Brace wall thickness
 R = Outer radius of chord
 r = Outer radius of brace
 Θ = Angle between chord and considered brace
 D = Outer diameter of chord

- d = Outer diameter of brace
- a = Gap (clear distance) between considered brace and nearest load-carrying brace measured along chord outer surface
- β = r/R
- γ = R/T
- g = a/D
- f_y = Yield stress
- Q_f = Factor
- Q_g = See table 6.1
- Q_u = See table 6.1
- $Q_{\beta d}$ = See table 6.1
- N = Design axial force in brace
- M_{IP} = Design in-plane bending moment in brace
- M_{OP} = Design out-of plane bending moment in brace
- N_k = Characteristic axial load capacity of brace (as governed by the chord strength)
- M_{OPk} = Characteristic out-of-plane bending moment capacity of brace (as governed by the chord strength)
- σ_{ax} = Design axial stress in chord
- σ_{IP} = Design in-plane bending stress in chord
- σ_{OP} = Design out-of-plane bending stress in chord

This section gives design formulae for simple tubular joints without overlap and without gussets, diaphragms or stiffeners. Tubular joints in a space frame structure shall satisfy:

$$N \leq \frac{N_k}{\gamma_m} \quad \text{where}$$

$$N_k = Q_u Q_f \frac{f_y T^2}{\sin \Theta}$$

Q_u is given in Table 6.1. Q_f is a factor to account for the nominal longitudinal stress in the chord.

$$Q_f = 1,0 - 0,03\gamma A^2 \quad \text{where}$$

$$A^2 = \frac{\sigma_{ax}^2 + \sigma_{IP}^2 + \sigma_{OP}^2}{0,64 f_y^2}$$

Table 6.1 Values for Q_u

Type of joint and geometry	Type of load in brace member		
	Axial	In-plane bending	Out-of-plane bending
T and Y	2,5+19 β	5,0 $\sqrt{\gamma\beta}$	$\frac{3,2}{1 - 0,81\beta}$
X	(2,7+13 β) Q_{β}		
K	0,90(2+21 β) Q_g		

$$Q_{\beta} = \frac{0,3}{\beta(1 - 0,833\beta)} \quad \text{for } \beta > 0,6$$

$$Q_{\beta} = 1,0 \quad \text{for } \beta \leq 0,6$$

$$Q_g = 1,8 - 0,1a / T \quad \text{for } \gamma \leq 20$$

$$Q_g = 1,8 - 4g \quad \text{for } \gamma > 20$$

but in no case shall Q_g be taken as less than 1,0.

When $\beta \geq 0,9$, Q_f is set to 1,0. This is also applicable for moment loading. For cases with tension in the chord, Q_f is set to 1.0. This is also applicable for moment loading.

The brace end moments shall be accounted for in the following cases:

- a) Out-of-plane bending moment when $\beta > 0,85$
- b) When the brace acts as a cantilever
- c) When the rotational stiffness of the connection is considered in the determination of effective buckling length, and / or the structural coefficient $\gamma_{mk} = 1.00$ for the beam-column design of the brace or chord. See Section 3.1.3.

The characteristic capacity of the brace subjected to in-plane bending moment shall be determined by:

$$M_{IPk} = Q_u Q_f \frac{df_y T^2}{\sin \Theta} \quad \text{where } Q_u \text{ is given in Table 6.1 and}$$

$$Q_f = 1,0 - 0,045\gamma A^2$$

The characteristic capacity of the brace subjected to out-of-plane bending moment shall be determined by:

$$M_{OPk} = Q_u Q_f \frac{df_y T^2}{\sin \Theta} \quad \text{where } Q_u \text{ is given in Table 6.1 and}$$

$$Q_f = 1,0 - 0,021\gamma A^2$$

For combined axial and bending loads in the brace, the following interaction equation should be satisfied:

$$\frac{N}{N_k} + \left(\frac{M_{IP}}{M_{IPk}} \right)^2 + \frac{M_{OP}}{M_{OPk}} \leq \frac{1}{\gamma_m}$$

For overlapping tubular joints without gussets, diaphragms, or stiffeners, the total load component normal to the chord, N_N , shall not exceed

$$N_N = \frac{N_k l_1}{\gamma_m l} \sin \Theta + \frac{2f_y t_w l_2}{\sqrt{3}\gamma_m}$$

where (see NPD fig. 3.10)

- l_1 = circumference for that portion of the brace in contact with the chord
(actual length)
- l = circumference of brace contact with chord, neglecting presence of
overlap
- N_k = characteristic axial load capacity of brace
- t_w = the lesser of the throat thickness of the overlapping weld or the thickness t of the
thinner brace
- l_2 = length as shown in NPD fig. 3.10

The above formula for the capacity of overlapping joints is valid only for K joints, where compression in a brace is essentially balanced by tension in brace(s) in the same side of the joint.

7 Tabulated results / TRACKs

This section presents a table with the various TRACKs available with respect to print out from the code check. Example prints and explanation to the information / heading given on the print out is given in Appendix A.

Table 7.1 Available TRACKs

TRACK no.	Description
0	Brief print of member utilizations (2 lines for each member) sorted with highest utilized members first
1	Based on TRACK 3 with additional information regarding stability factors and capacities
2	Simple print of stresses, incl von Mises stress
3	Brief print of member utilizations (2 lines for each member)
9	Comprehensive print with detailed information about member and member utilization(one page for each member)
99	Used in connection with tubular joint check according to NPD. This TRACK identifies tubular joints to be checked and classifies all members entering the joint as T connection
98	Used in connection with tubular joint check according to NPD. This TRACK performs the joint capacity check
49	Prints member end forces for members entering each joint (at the end of the member connected to the joint)
31	Prints maximum and minimum member end forces (axial force defines max and min) at member end 1
32	Prints maximum and minimum member end forces (axial force defines max and min) at member end 2

Appendix A

Tracks for member code checking

TRACK = 0.0

NS3472 (VERSION 96016.00)
UNITS ARE KNS AND METE

MEMB	FX TABLE	MYs MZs	MYm MZm	MYe MZe	MYb MZb	RATIO COND	LOAD DIST
21	154.18 C	1.8	.5	.7	.8	1.00	11
FAIL	PIP 300X10	-185.5	-98.5	10.3	116.0	VMIS	.00
84	.28 T	-211.1	313.7	191.6	311.7	1.00	11
	TUBRHSBEAM	-764.7	-1666.4	777.1	1666.4	VMIS	.57
99	783.66 C	.4	.2	.0	.2	.92	11
	PRISHANGOFF	2480.6	1240.8	.0	1488.8	STAB	.00
111	299.78 T	-13.2	-17.5	21.8	18.4	.86	11
	PIP 600X15	-426.9	262.5	-951.8	400.3	VMIS	3.46
125	310.50 T	-21.8	-24.3	26.9	24.8	.82	11
	PIP 600X30	951.8	1350.4	-1749.0	1430.1	VMIS	2.00
133	2690.57 C	.0	5.4	-10.8	6.5	.74	11
	PIP 600X30	.0	589.8	-1179.7	707.8	VMIS	1.15
31	164.29 C	.8	-.6	1.9	.8	.71	11
	PIP 300X10	13.0	71.3	-128.6	82.9	VMIS	2.17
123	699.97 C	-7.6	-6.5	5.4	6.7	.62	11
	PIP 600X15	-618.4	-541.7	465.0	557.0	VMIS	.00

SYMBOL	DESCRIPTION TRACK 0.0	UNIT
MEMB	- Member number	
FX	- Axial force in the member (T= tension, C=compression)	kN
MYs	- Start moment about y-axis	kNm
MYm	- Mid moment about y-axis	kNm
MYe	- End moment about y-axis	kNm
MYb	- Buckling moment about y-axis	kNm
RATIO	- Interaction ratio	
LOAD	- The critical load case number	
TABLE	- Section type (HE, IPE, TUBE etc.)	
MZs	- Start moment about z-axis	kNm
MZm	- Mid moment about z-axis	kNm
MZe	- End moment about z-axis	kNm
MZb	- Buckling moment about z-axis	kNm
COND	- Critical condition	
DIST	- Distance from the start of the member to the critical section	m

NB! Myb and Mzb are the design moments used for max unity ratio.

TRACK = 1.0

NS3472 (VERSION 96016.00)
UNITS ARE KNS AND METE

MEMB	FX TABLE	MYs MZs	MYm MZm	MYe MZe	MYb MZb	RATIO COND	LOAD DIST
111	299.78 T PIP 600X15	-13.2 -426.9	-17.5 262.5	21.8 -951.8	18.4 400.3	.86 VMIS	11 3.46

CURVE St A Wk A Beta Z .80 Beta Y .80 FYLD= 345. N/MM2							
NKYD=.827E+4 KN NKZD=.827E+4 KN NEYD=.271E+6 KN NEZD=.271E+6 KN							
MYD =.118E+4 KNM MZD =.118E+4 KNM MVD =.118E+4 KNM LAMBDA= 13.37							
STRONG IR = .000 WEAK IR = .000 VON MISES = .860 LATBUCK= .786							

112	377.68 T PIP 600X15	-13.8 224.1	-11.8 -17.7	9.8 259.5	12.2 103.8	.28 VMIS	13 2.83

CURVE St A Wk A Beta Z .80 Beta Y .80 FYLD= 345. N/MM2							
NKYD=.827E+4 KN NKZD=.827E+4 KN NEYD=.405E+6 KN NEZD=.405E+6 KN							
MYD =.118E+4 KNM MZD =.118E+4 KNM MVD =.118E+4 KNM LAMBDA= 10.94							
STRONG IR = .000 WEAK IR = .000 VON MISES = .276 LATBUCK= .185							

113	11.61 T PIP 600X15	-18.4 -43.3	-16.4 132.9	14.3 -309.1	16.8 168.2	.27 LATB	11 3.46

CURVE St A Wk A Beta Z .80 Beta Y .80 FYLD= 345. N/MM2							
NKYD=.827E+4 KN NKZD=.827E+4 KN NEYD=.271E+6 KN NEZD=.271E+6 KN							
MYD =.118E+4 KNM MZD =.118E+4 KNM MVD =.118E+4 KNM LAMBDA= 13.37							
STRONG IR = .000 WEAK IR = .000 VON MISES = .267 LATBUCK= .275							

114	366.02 C PIP 600X15	-13.2 -224.8	-11.3 120.9	9.3 -466.5	11.6 190.0	.45 VMIS	11 2.83

CURVE St A Wk A Beta Z .80 Beta Y .80 FYLD= 345. N/MM2							
NKYD=.827E+4 KN NKZD=.827E+4 KN NEYD=.405E+6 KN NEZD=.405E+6 KN							
MYD =.118E+4 KNM MZD =.118E+4 KNM MVD =.118E+4 KNM LAMBDA= 10.94							
STRONG IR = .206 WEAK IR = .206 VON MISES = .452 LATBUCK= .000							

SYMBOL	DESCRIPTION TRACK 1.0	UNIT
CURVE St	- Buckling curve about <u>S</u> trong axis	
CURVE Wk	- Buckling curve about <u>W</u> eak axis	
Beta Z	- Buckling length factor about z-axis	
Beta Y	- Buckling length factor about y-axis	
FYLD	- Allowable yield strength	N/mm ²
NKYD	- Factored buckling strength/ resistance about y-axis	kN
NKZD	- Factored buckling strength/ resistance about z-axis	kN
NKD	- Axial capacity	kN
NEYD	- Euler buckling resistance for compression members about y- axis	kN
NEZD	- Euler buckling resistance for compression members about z-axis	kN
MYD	- Moment capacity about y-axis	kNm
MZD	- Moment capacity about z-axis	kNm
MVD	- Lateral buckling moment	kNm
LAMBDA	- $\lambda = \frac{L_k}{i}$	
STRONG IR	- Interaction Ratio for buckling about strong axis	
WEAK IR	- Interaction Ratio for buckling about weak axis	
VON MISES	- Interaction Ratio for von Mises	
LATBUCK	- Interaction Ratio for lateral buckling	

TRACK = 2.0

NS3472 (VERSION 96016.00)
 UNITS ARE mm AND N

MEMB	Sx TABLE	Sby Ty	Sbz Tz	Stot Tto	Spmx Spmn	Svm POINT	LOAD DIST
111	10.87 T PIP 600X15	242.0 28.9	.0 .0	252.9 .3	.0 .0	257.90 3	11 3.46
112	13.70 T PIP 600X15	66.0 12.4	.0 .0	79.7 .3	.0 .0	82.70 3	13 2.83
113	.42 T PIP 600X15	78.7 7.4	.0 .0	79.1 .2	.0 .0	80.18 3	11 3.46
114	13.28 C PIP 600X15	118.6 17.7	.0 .0	131.9 .3	.0 .0	135.52 3	11 2.83
115	68.71 C PIP 200X8	18.8 1.0	.0 .0	87.5 2.3	.0 .0	87.66 3	11 .00
116	66.13 T PIP 200X8	37.9 1.1	.0 .0	104.0 10.3	.0 .0	105.91 3	11 5.13
117	63.29 T PIP 200X8	47.5 1.7	.0 .0	110.8 .4	.0 .0	110.84 3	11 .00
118	85.64 C PIP 200X8	39.8 1.2	.0 .0	125.4 .2	.0 .0	125.43 3	11 6.20
119	90.54 C PIP 200X8	21.4 1.0	.0 .0	111.9 .6	.0 .0	111.96 3	14 .00
120	94.89 T PIP 200X8	24.7 1.2	.0 .0	119.6 .3	.0 .0	119.61 3	14 5.23
121	79.98 C PIP 200X8	43.7 1.3	.0 .0	123.6 .2	.0 .0	123.68 3	11 6.20

SYMBOL**DESCRIPTION TRACK 2.0****UNIT**

MEMB	- Member number	
Sx	- Axial stress in the member (T= tension, C=compression)	N/mm ²
Sby	- Stress from moment about y-axis	N/mm ²
Sbz	- Stress from moment about z-axis	N/mm ²
Stot	- Sum of Sx + Sby + Sbz	N/mm ²
Spmx	- Currently not in use	
Spmn	- Currently not in use	
Svm	- von Mises stress	N/mm ²
Ty	- Stress from shear force in y direction	N/mm ²
Tz	- Stress from shear force in z direction	N/mm ²
Tto	- Total shear stress used in von Mises calculation	N/mm ²
TABLE	- Section type (HE, IPE, TUBE etc.)	
POINT	- Location in cross section with max von Mises stress	
LOAD	- Governing load condition	
DIST	- Distance from the start of the member to the critical section	m

Note:

Do not use TRACK = 2.0 in connection with the SELECT OPTIMIZED or SELECT MEMBER / ALL commands.

TRACK = 3.0

NS3472 (VERSION 96016.00)
UNITS ARE KNS AND METE

MEMB	FX TABLE	MYs MZs	MYm MZm	MYe MZe	MYb MZb	RATIO COND	LOAD DIST
111	299.78 T PIP 600X15	-13.2 -426.9	-17.5 262.5	21.8 -951.8	18.4 400.3	.86 VMIS	11 3.46
112	377.68 T PIP 600X15	-13.8 224.1	-11.8 -17.7	9.8 259.5	12.2 103.8	.28 VMIS	13 2.83
113	11.61 T PIP 600X15	-18.4 -43.3	-16.4 132.9	14.3 -309.1	16.8 168.2	.27 LATB	11 3.46
114	366.02 C PIP 600X15	-13.2 -224.8	-11.3 120.9	9.3 -466.5	11.6 190.0	.45 VMIS	11 2.83
115	331.57 C PIP 200X8	-4.1 -.5	.3 1.0	-2.0 -2.5	1.7 1.3	.33 STAB	11 .00
116	319.14 T PIP 200X8	-1.9 -.4	-.6 3.7	-3.3 -7.8	1.4 4.5	.35 VMIS	11 5.13
117	305.44 T PIP 200X8	.0 10.6	-.3 -.8	.5 9.9	.3 5.3	.37 VMIS	11 .00
118	413.31 C PIP 200X8	.1 4.4	-.3 -1.1	.6 8.8	.3 4.6	.50 STAB	11 6.20
119	436.94 C PIP 200X8	-4.5 -1.6	.2 .0	-2.1 -1.7	1.9 .7	.41 STAB	14 .00
120	457.92 T PIP 200X8	-3.7 1.7	-.7 -.2	-5.1 2.1	2.0 .8	.40 VMIS	14 5.23
121	385.99 C PIP 200X8	-.2 6.0	-.1 -.8	.0 9.7	.1 5.0	.48 STAB	11 6.20

SYMBOL	DESCRIPTION TRACK 3.0	UNIT
MEMB	- Member number	
FX	- Axial force in the member (T= tension, C=compression)	kN
MYs	- Start moment about y-axis	kNm
MYm	- Mid moment about y-axis	kNm
MYe	- End moment about y-axis	kNm
MYb	- Buckling moment about y-axis	kNm
RATIO	- Interaction ratio	
LOAD	- The critical load case number	
TABLE	- Section type (HE, IPE, TUBE etc.)	
MZs	- Start moment about z-axis	kNm
MZm	- Mid moment about z-axis	kNm
MZe	- End moment about z-axis	kNm
MZb	- Buckling moment about z-axis	kNm
COND	- Critical condition	
DIST	- Distance from the start of the member to the critical section	m

TRACK 3: Member results sorted by member number.

TRACK = 9.0

Member in tension:

DETAILS FOR CODECHECK ACCORDING TO NS3472

(VERSION 96016.00)

MEMBER NO : 111
 MEMBER TYPE : PIPE SECTION PIP 600X15
 GOVERNING LOADCASE : 11

MEMBER PROPERTY	UNITS	CM
Ax	:	275.7
Ay	:	137.8
Az	:	137.8
Ix	:	236010.1
Iy	:	118005.0
Iz	:	118005.0
iy	:	20.7
iz	:	20.7
Sy	:	3933.5
Sz	:	3933.5
Iw	:	.0
Lw	:	345.9

MATERIAL DATA	UNITS	NEWTON	MMS
E	:	204960.	Gamma : 1.150
Fy	:	344.966	Fd : 299.970

FORCES	UNITS	KNEWTON	METERS
Fx	:	299.78	T
Msz	:	-426.86	Msy : -13.176
Mmz	:	262.48	Mmy : -17.508
Mez	:	-951.82	Mey : 21.840

LATERAL BUCKLING

Mlatbuck: 670.43
 Mvd : 1179.90
 IRtot : .786

YIELD CHECK

STRESS : NEW	MMS	FORCES: KNEW	METERS
STRESS AT POINT :	3	FORCES AT SECTION	3.459
sigax	:	10.873	Fx : 299.778 T
sigb	:	242.016	Fy : 398.539
tau	:	28.914	Fz : 2.505
tors	:	.305	Mx : 2.398
sige	:	257.904	My : 21.838
IR	:	.860	Mz : -951.695

Governing interaction ratio .860

Member in compression:

DETAILS FOR CODECHECK ACCORDING TO NS3472
(VERSION 96016.00)

MEMBER NO : 114
MEMBER TYPE : PIPE SECTION PIP 600X15
GOVERING LOADCASE : 11

MEMBER PROPERTY		UNITS CM	
Ax	: 275.7	iy	: 20.7
Ay	: 137.8	iz	: 20.7
Az	: 137.8	Sy	: 3933.5
Ix	: 236010.1	Sz	: 3933.5
Iy	: 118005.0	Iw	: .0
Iz	: 118005.0	Lw	: 282.9

MATERIAL DATA		UNITS NEWTON MMS	
E	: 204960.	Gamma	: 1.150
Fy	: 344.966	Fd	: 299.970
lamfy	: 76.577	Gamma mk:	1.000

BUCKLING PARAMETERS UNITS KNEWTON METERS

STRONG AXIS		WEAK AXIS		LATERAL BUCKLING	
L	: 2.829	L	: 2.829	L	: 2.829
beta	: .800	beta	: .800	ny	: 1.000
lambda:	10.939	lambda:	10.939	n	: 1.500
lambb	: .143	lambb	: .143	Mvd	: 1179.903
curve	: A	curve	: A		
Fk/Fy	: 1.000	Fk/Fy	: 1.000		
Nkd	: 8269.398	Nkd	: 8269.398		
NEd	: 405254.700	NEd	: 405254.700		
Md	: 1179.903	Md	: 1179.903		
KE	: 1.000	1/KE	: 1.000		
Factor:	.999	Factor:	.999		

FORCES UNITS KNEWTON METERS

STRONG AXIS		WEAK AXIS	
Fx	: 366.025 C	Fx	: 366.025 C
Ms	: -224.840	Ms	: -13.205
Mm	: 120.851	Mm	: -11.253
Me	: -466.542	Me	: 9.300
beta	: -.482	beta	: .704
m	: .407	m	: .882
Mb	: 189.989	Mb	: 11.643
IRx	: .044	IRx	: .044
IRm	: .161	IRm	: .161
IRtot	: .206	IRtot	: .206

YIELD CHECK

STRESS : NEW MMS		FORCES: KNEW METERS	
STRESS AT POINT :	3	FORCES AT SECTION	2.829
sigax	: 13.276	Fx	: 366.025 C
sigb	: 118.618	Fy	: 244.366
tau	: 17.729	Fz	: -1.380
tors	: .260	Mx	: 2.044
sige	: 135.525	My	: 9.299
IR	: .452	Mz	: -466.480

Governing interaction ratio .452

Member in compression (pipe - NPD):

DETAILS FOR CODECHECK ACCORDING TO NPD94

```

                                (VERSION 96016.00)
MEMBER NO                       :      1
MEMBER TYPE                     : PIPE      762x  19 mm
GOVERNING LOADCASE              :      2

UNITS [properties: cm] [stesses:new mms] [forces:kn me]
-- PROPERTIES --
D/t   :      40.0      Iy Iz : 306983.8      Ly   :      3074.8
Ax    :      444.6      Sy Sz :      8057.3      Lz   :      3074.8
Ay Az :      222.3      iy iz :      26.3      By   :      .7
Ix    : 613967.7      Z     : 124263.4      Bz   :      .6

-- MATERIAL --
E     : 209979.      Fy   :      366.      Gamma m :      1.150
lamfy :      91.      Fd   :      318.      Gamma mk:      1.000

-- SHELL BUCKLING --
-- Section npd 3.4.6.1 --
fea : 2984.5 feb : 3076.4 fet : 528.2 fep : 131.2
-- Section npd 3.4.4.1 --
La : .350 Lb : .345 Lt : .632 Lp : 1.670
Ga : 1.047 Gb : 1.045 Gt : 1.135 Gp : 1.250
Fk : 365.952
-- Section npd 3.4.7 --
Sigj: 278.7 SECT: 1.0 Irshell .876
-- Section npd 3.4.9.2 --
Bendingmoment stress in 3.4.4.1 increased by dsigb : 91.430
d/t 40.0 > 12.0 interaction npd 3.2.3 a
-- BEAM COLUMN BUCKLING --
-- Section npd 3.4.4.1 --
La : .350 Lb : .345 Lt : .632 Lp : 1.670
Ga : 1.047 Gb : 1.045 Gt : 1.135 Gp : 1.250
Fa : 249.6 Sect: 1.0
-- Section ns3472 5.4.1 --
lbz : 67.872 lbbz: .745 crvz: A FkFy z: .826
lby : 78.404 lbby: .860 crvy: A FkFy y: .759
-- Section npd 3.2.2 --
SIGa : 162.6 SIGby : 8.2 SIGbz : 32.2
fE : 337.131
B : 1.932 By : 1.932 Bz : 1.566
SIGb* : 26.3 SIGbuc: 266.4 Irb : 1.227
-- Section npd 3.2.2.1 (ns3472 5.4.2) --
z axis y axis
Fx : 7231.114 C Fx : 7231.114 C
Ms : -87.503 Ms : 28.677
Mm : -154.416 Mm : 36.766
Me : -297.803 Me : 87.503
beta : -.294 beta : -.328
m : .482 m : .469
Mb : 259.566 Mb : 66.179
-- Section npd 3.1.2 --
Stress at point : 3 Forces at section 30.748
sigax : 162.614 Fx : 7231.114 C
sigb : 38.519 Fy : 46.293
tau : 2.156 Fz : 12.386
tors : .000 Mx : .000
sigp : 30.166 My : 87.491
sige : 187.912 Mz : -297.764
Irj fy : .590 HSpres: 1.508

Governing interaction ratio 1.227

```


Tracks for joint capacity code checking**TRACK = 99**

\$JOINT	BRACE	CHORD	D	T	d	t	GAP	FYc	FYb	THETA	TW	THETAT	JTYPE
1020	1016	1015	420.	15.	400.	15.	0.	340.	340.	90.	0.	0.	TY
1020	1016	1017	420.	15.	400.	15.	0.	340.	340.	90.	0.	0.	TY
2010	1016	1010	500.	20.	400.	15.	0.	340.	340.	45.	0.	0.	TY
2010	1016	2010	500.	20.	400.	15.	0.	340.	340.	45.	0.	0.	TY
2000	1017	1000	500.	20.	420.	15.	0.	340.	340.	45.	0.	0.	TY
2000	1017	2000	500.	20.	420.	15.	0.	340.	340.	45.	0.	0.	TY
1020	1018	1015	420.	15.	400.	15.	0.	340.	340.	90.	0.	0.	TY
1020	1018	1017	420.	15.	400.	15.	0.	340.	340.	90.	0.	0.	TY
2000	2015	1000	500.	20.	400.	15.	0.	340.	340.	40.	0.	0.	TY
2000	2015	2000	500.	20.	400.	15.	0.	340.	340.	40.	0.	0.	TY
3010	2015	2010	500.	20.	400.	15.	0.	340.	340.	40.	0.	0.	TY
3010	2015	3010	500.	20.	400.	15.	0.	340.	340.	40.	0.	0.	TY
3010	3015	2010	500.	20.	400.	15.	0.	340.	340.	40.	0.	0.	TY
3010	3015	3010	500.	20.	400.	15.	0.	340.	340.	40.	0.	0.	TY
2000	2005	1000	500.	20.	400.	10.	0.	340.	340.	90.	0.	0.	TY
2000	2005	2000	500.	20.	400.	10.	0.	340.	340.	90.	0.	0.	TY
2010	2005	1010	500.	20.	400.	10.	0.	340.	340.	90.	0.	0.	TY
2010	2005	2010	500.	20.	400.	10.	0.	340.	340.	90.	0.	0.	TY
3000	3005	2000	500.	20.	400.	10.	0.	340.	340.	90.	0.	0.	TY
3000	3005	3000	500.	20.	400.	10.	0.	340.	340.	90.	0.	0.	TY
3010	3005	2010	500.	20.	400.	10.	0.	340.	340.	90.	0.	0.	TY
3010	3005	3010	500.	20.	400.	10.	0.	340.	340.	90.	0.	0.	TY
2210	1215	1210	500.	20.	400.	15.	0.	340.	340.	45.	0.	0.	TY
2210	1215	2210	500.	20.	400.	15.	0.	340.	340.	45.	0.	0.	TY
2210	2215	1210	500.	20.	400.	15.	0.	340.	340.	40.	0.	0.	TY
2210	2215	2210	500.	20.	400.	15.	0.	340.	340.	40.	0.	0.	TY
3200	2215	2200	500.	20.	400.	15.	0.	340.	340.	40.	0.	0.	TY
3200	2215	3200	500.	20.	400.	15.	0.	340.	340.	40.	0.	0.	TY
3200	3215	2200	500.	20.	400.	15.	0.	340.	340.	40.	0.	0.	TY
3200	3215	3200	500.	20.	400.	15.	0.	340.	340.	40.	0.	0.	TY
2200	2205	1200	500.	20.	400.	10.	0.	340.	340.	90.	0.	0.	TY
2200	2205	2200	500.	20.	400.	10.	0.	340.	340.	90.	0.	0.	TY

TRACK = 98

NPD 94 TUBULAR JOINT CHECK (VERSION 96016.00)
UNITS ARE mm

JOINT	CHORD	Dc	Tc	BRACE	db	tb	TYPE	LOAD	RATIO
1020	1015	420.0	15.0	1016	400.0	15.0	TY	3	.143
1020	1017	420.0	15.0	1016	400.0	15.0	TY	3	.143
2010	1010	500.0	20.0	1016	400.0	15.0	TY	3	.049
2010	2010	500.0	20.0	1016	400.0	15.0	TY	3	.048
2000	1000	500.0	20.0	1017	420.0	15.0	TY	3	.548
2000	2000	500.0	20.0	1017	420.0	15.0	TY	3	.586
1020	1015	420.0	15.0	1018	400.0	15.0	TY	3	.099
1020	1017	420.0	15.0	1018	400.0	15.0	TY	3	.099
2000	1000	500.0	20.0	2015	400.0	15.0	TY	3	.450
2000	2000	500.0	20.0	2015	400.0	15.0	TY	3	.481
3010	2010	500.0	20.0	2015	400.0	15.0	TY	3	.527
3010	3010	500.0	20.0	2015	400.0	15.0	TY	3	.498
3010	2010	500.0	20.0	3015	400.0	15.0	TY	3	.338
3010	3010	500.0	20.0	3015	400.0	15.0	TY	3	.320
2000	1000	500.0	20.0	2005	400.0	10.0	TY	3	.107
2000	2000	500.0	20.0	2005	400.0	10.0	TY	3	.113
2010	1010	500.0	20.0	2005	400.0	10.0	TY	3	.177
2010	2010	500.0	20.0	2005	400.0	10.0	TY	3	.170
3000	2000	500.0	20.0	3005	400.0	10.0	TY	3	.168
3000	3000	500.0	20.0	3005	400.0	10.0	TY	3	.167
3010	2010	500.0	20.0	3005	400.0	10.0	TY	3	.183
3010	3010	500.0	20.0	3005	400.0	10.0	TY	3	.174
2210	1210	500.0	20.0	1215	400.0	15.0	TY	3	.945
2210	2210	500.0	20.0	1215	400.0	15.0	TY	3	.518
2210	1210	500.0	20.0	2215	400.0	15.0	TY	3	1.146
2210	2210	500.0	20.0	2215	400.0	15.0	TY	3	.617
3200	2200	500.0	20.0	2215	400.0	15.0	TY	3	.575
3200	3200	500.0	20.0	2215	400.0	15.0	TY	3	.579
3200	2200	500.0	20.0	3215	400.0	15.0	TY	3	.232
3200	3200	500.0	20.0	3215	400.0	15.0	TY	3	.234
2200	1200	500.0	20.0	2205	400.0	10.0	TY	3	.183
2200	2200	500.0	20.0	2205	400.0	10.0	TY	3	.177
2210	1210	500.0	20.0	2205	400.0	10.0	TY	3	1.402
2210	2210	500.0	20.0	2205	400.0	10.0	TY	3	.262
3200	2200	500.0	20.0	3205	400.0	10.0	TY	3	.210
3200	3200	500.0	20.0	3205	400.0	10.0	TY	3	.212
3210	2210	500.0	20.0	3205	400.0	10.0	TY	3	.223
3210	3210	500.0	20.0	3205	400.0	10.0	TY	3	.223
2000	1000	500.0	20.0	1315	400.0	15.0	TY	3	.522
2000	2000	500.0	20.0	1315	400.0	15.0	TY	3	.558
2000	1000	500.0	20.0	2315	400.0	15.0	TY	3	.463
2000	2000	500.0	20.0	2315	400.0	15.0	TY	3	.495
3200	2200	500.0	20.0	2315	400.0	15.0	TY	3	.519
3200	3200	500.0	20.0	2315	400.0	15.0	TY	3	.523
3200	2200	500.0	20.0	3315	400.0	15.0	TY	3	.309
3200	3200	500.0	20.0	3315	400.0	15.0	TY	3	.311
2200	1200	500.0	20.0	2305	400.0	10.0	TY	3	.259
2200	2200	500.0	20.0	2305	400.0	10.0	TY	3	.238
2000	1000	500.0	20.0	2305	400.0	10.0	TY	3	.154
2000	2000	500.0	20.0	2305	400.0	10.0	TY	3	.171
3200	2200	500.0	20.0	3305	400.0	10.0	TY	3	.163
3200	3200	500.0	20.0	3305	400.0	10.0	TY	3	.164
3000	2000	500.0	20.0	3305	400.0	10.0	TY	3	.159
3000	3000	500.0	20.0	3305	400.0	10.0	TY	3	.159

Special prints (not code check)**TRACK = 49**NS3472 JOINT OUTPUT (VERSION 95015.01.)
UNITS ARE KNS AND METE

JOINT	LOAD	MEMBER	FX	FY	FZ	MZ	MY	MZ
1	12	1	46.8C	142.0	156.0	72.3	38.8	-31.7
		8	9.5C	51.6	-13.0	4.4	-28.2	-109.4
		107	193.5T	33.8	-146.5	10.6	-181.7	27.3
2	12	1	46.8C	-140.7	-156.0	-72.3	88.8	-83.9
		2	110.8C	22.7	-.1	-.2	-11.8	82.0
		9	156.1T	118.0	-64.0	1.9	-76.9	-72.5
3	12	2	110.8C	22.2	.1	.2	11.4	-82.8
		3	47.3C	-140.1	-158.7	-68.3	-86.7	84.2
		10	158.6T	117.8	63.5	-1.4	75.3	-68.0
4	12	3	47.3C	141.3	158.7	68.3	-43.1	30.9
		4	19.1C	51.6	13.0	-4.4	26.9	-107.4
		108	192.9T	-34.3	-139.6	-16.2	-175.6	-26.5
5	12	4	19.1C	-44.8	-13.0	4.4	30.3	-104.8
		5	1.2T	47.4	119.8	-69.3	43.3	-16.8
		109	92.2C	-11.8	-138.9	-13.0	-174.1	12.4
6	12	5	1.2T	-48.6	-119.8	69.3	54.7	-22.4
		6	64.6T	-22.6	278.3	278.5	149.3	21.1
		10	158.6T	-71.2	-63.5	1.4	204.0	-347.8
7	12	6	64.6T	-22.3	279.4	279.2	-151.5	-20.5
		7	.7T	-49.1	-123.3	65.1	-53.0	22.4
		9	156.1T	-71.4	64.0	-1.9	-204.5	-344.3
8	12	7	.7T	47.8	123.3	-65.1	-47.8	17.2
		8	9.5C	-44.8	13.0	-4.4	-29.2	-102.8
		110	92.6C	12.4	-132.8	18.7	-167.8	-12.8
9	12	11	218.4T	81.6	27.7	75.1	2.3	-94.9
		18	101.5T	48.6	-3.1	-7.8	-8.7	-101.8
		107	200.7T	-33.8	146.5	-10.6	-201.5	-115.7
		111	549.0T	-169.9	3.6	3.4	20.5	221.7
		115	333.3C	.3	2.2	-.2	3.9	-1.0
		117	40.2C	1.1	.0	.1	.1	-2.5

TRACK = 31

MAX MIN OUTPUT FOR END NO: 1
 UNITS ARE KNS AND METE

MEMBER	LOAD	FX	FY	FZ	MZ	MY	MZ
1	11	67.6C	153.8	155.2	71.9	38.2	-61.3
	16	-116.3T	60.0	-3.7	.1	1.2	-181.9
	11	67.6C	153.8	155.2	71.9	38.2	-61.3
	10	.9C	.1	.0	.0	.0	-.1
	13	48.0C	142.0	156.0	72.3	38.8	-31.6
	16	-116.3T	60.0	-3.7	.1	1.2	-181.9
	13	48.0C	142.0	156.0	72.3	38.8	-31.6
	1	3.0C	36.3	.3	-.2	-.1	9.0
	14	1.1C	96.6	77.1	41.7	39.2	-58.8
	3	-12.4T	29.1	45.2	24.7	-23.1	30.2
	4	-81.9T	7.5	-2.6	.1	-.6	103.1
	16	-116.3T	60.0	-3.7	.1	1.2	-181.9

TRACK = 32

MAX MIN OUTPUT FOR END NO: 2
 UNITS ARE KNS AND METE

MEMBER	LOAD	FX	FY	FZ	MZ	MY	MZ
1	11	67.6C	-152.5	-155.2	-71.9	88.7	-64.0
	16	-116.3T	-58.7	3.7	-.1	-4.2	133.4
	10	.9C	-.1	.0	.0	.0	.1
	11	67.6C	-152.5	-155.2	-71.9	88.7	-64.0
	16	-116.3T	-58.7	3.7	-.1	-4.2	133.4
	13	48.0C	-140.8	-156.0	-72.3	88.8	-84.0
	1	3.0C	-35.4	-.3	.2	-.2	20.3
	13	48.0C	-140.8	-156.0	-72.3	88.8	-84.0
	13	48.0C	-140.8	-156.0	-72.3	88.8	-84.0
	2	14.7C	-56.0	-91.8	-42.8	-52.3	31.6
	16	-116.3T	-58.7	3.7	-.1	-4.2	133.4
	4	-81.9T	-7.5	2.6	-.1	2.7	-97.0

8 References

- [1] NS 3472 3.utg. 2001
Prosjektering av stålkonstruksjoner
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- [2] STAAD.Pro Technical Reference Manual, Release 2002
- [3] NS 3472 1.utg. 1973
Prosjektering av stålkonstruksjoner
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- [4] Roark & Young`s 5th edition
- [5] NPD utg. 1994
Veiledning om utforming, beregning og dimensjonering av stålkonstruksjoner. Sist
endret 1. oktober 1993.
- [6] NS 3472 2.utg.1984
Prosjektering av stålkonstruksjoner
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